NASA TECHNICAL NOTE



NASA TN D-6402

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SUBCOOLED- AND NET-BOILING
HEAT TRANSFER TO LOW-PRESSURE
WATER IN ELECTRICALLY HEATED TUBES

by James R. Stone

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION • WASHINGTON, D. C. • JULY 1971

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1. Report No.	-	2. Government Accessi	on No.	3. Recipient's Catalog	No.
NASA TN D-6402	1				
4. Title and Subtitle	T DOITE	NO TIES OF ORDANIC	TEED TO LOW	5. Report Date July 1971	
SUBCOOLED - AND NE			i	6. Performing Organiza	ation Code
PRESSURE WATER IN	ELECTR	ICALLY HEATE	D TUBES		
7. Author(s)				8. Performing Organiza	ition Report No.
James R. Stone				E-6207	
				10. Work Unit No.	
9. Performing Organization Name and				120-27	
Lewis Research Center			Ī	11. Contract or Grant	No.
National Aeronautics ar	nd Space	Administration			
Cleveland, Ohio 44135				13. Type of Report an	d Period Covered
2. Sponsoring Agency Name and Ac				Technical No	ote
National Aeronautics ar	-	Administration		14. Sponsoring Agency	Code
Washington, D.C. 205	46				
5. Supplementary Notes					
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7. Key Words (Suggested by Autho	r(s))		18. Distribution Statement	t	
Boiler	Net-qu	ality boiling	Unclassified - 1	unlimited	
Heat transfer	Experi	mental			
Nonequilibrium	Correl	ations			
Subcooled boiling					
9. Security Classif, (of this report)		20. Security Classif. (c	of this page)	21. No. of Pages	22. Price*
Unclassified			lassified	37	\$3.00

^{*} For sale by the National Technical Information Service, Springfield, Virginia 22151

SUBCOOLED- AND NET-BOILING HEAT TRANSFER TO LOW-PRESSURE WATER IN ELECTRICALLY HEATED TUBES

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SUMMARY

Experimental data are presented on subcooled and net-quality boiling heat transfer to water flowing vertically upward in 0.584- and 1.219-centimeter inside diameter tubes with uniform heat flux. Axial inner-wall-temperature distributions are tabulated for mass velocities from 0.67 to 141 kg/(sec)(m 2), heat fluxes from 43.8 to 11 400 kW/m 2 , exit pressures from 24 to 690 kN/m 2 abs, exit qualities up to 0.65, and liquid subcoolings as high as 151 K. Since no satisfactory correlations are available for the full range of test conditions, these experimental data over a wide range of test conditions should be useful to the designer.

The subcooled-boiling data are compared with some existing correlations. These correlations give the heat flux as a function of wall temperature minus saturation temperature. The data show only approximate agreement with the correlations, due probably to the effects of mass velocity, local subcooling, and distance from the inception of boiling. But the correlations are in the range of the data and may be useful for some applications.

Plots of net-boiling heat-transfer coefficients against quality do not agree with any existing correlation; the data show that, for constant heat flux, mass velocity, and quality, the local heat-transfer coefficient increases as pressure increases, whereas the correlations predict the opposite.

Heat-transfer coefficients, based on a wall-to-liquid temperature difference corrected for nonequilibrium, vary less with quality and show more consistent trends than do heat-transfer coefficients based on either wall-to-bulk or wall-to-saturation temperature differences. But no presently available nonequilibrium model appears to be valid over the entire range of liquid subcoolings of this study. Thus, a general heat-transfer correlation must be preceded by a nonequilibrium model valid for all subcoolings encountered.

INTRODUCTION

An understanding of forced-flow boiling phenomena is necessary for the rational design of Rankine-cycle power systems, especially those for use in space, where compactness is important. Boiling, with its high heat-transfer coefficients, is also applicable to cooling problems where high heat fluxes are involved, such as in rocket-nozzle cooling. To obtain efficient, compact space power systems, high fluid temperatures are required. These high temperatures can be obtained at relatively low pressures by using alkali metals as working fluids. With the exception of liquid thermal conductivity, water has physical properties similar to the alkali metals. Since experiments on the boiling of alkali metals are difficult and expensive to perform, and since water itself may be of interest for cooling applications, a series of experiments on heat transfer and pressure drop for water boiling in tubes has been done at the NASA Lewis Research Center (refs. 1 to 5).

Although there have been numerous studies of boiling heat transfer, there is still no generally applicable means of prediction available for high-density-ratio fluids such as alkali metals and low-pressure water. This is especially true of the subcooled-boiling regime. Experimental results for high-heat-flux subcooled boiling of low-pressure water in 0.584-centimeter inside diameter tubes were presented in reference 1. In fully developed subcooled boiling, it was found that the wall temperature was nearly independent of fluid bulk temperature and mass velocity for a given pressure level and heat flux (the heat flux being in the range of about 1600 to 9100 kW/m 2). Existing semiempirical correlations (refs. 6 to 8) did not satisfactorily predict the heat transfer at low pressures.

It is the objective of this study to obtain subcooled- and net-boiling heat-transfer data over a wide range of test variables, including the range covered in the heat-exchanger boiling studies (ref. 2). The range of interest is from the inception of boiling up to, but not including, the heat-transfer transition often called 'burnout'. This regime or series of regimes is often termed 'nucleate boiling'; however, no such terminology is used herein since this range may include other regimes, such as evaporation from the interface with or without nucleation. The correlations of references 6 to 9 for subcooled-boiling heat transfer and references 10 to 14 for net-boiling heat transfer, based primarily on annular-flow models, are compared with the experimental data to determine whether or not any of them are applicable over the range of the experiments.

In order to obtain heat-transfer data under conditions comparable to those of the heat-exchanger boiling studies (ref. 2), the present heat-transfer studies employ 1.219-as well as 0.584-centimeter inside diameter tubes. The experimental test sections of this study are made of Inconel X tubing having a 0.025-centimeter-thick wall. The water flows vertically upward through the straight, circular tube. No inserts are used. Exit qualities as high as 0.65 are obtained.

APPARATUS

Figure 1 shows a schematic diagram of the test apparatus. This is essentially the same test apparatus as that used in reference 1, wherein it is described in more detail. The system is designed to operate at a maximum pressure of 1700 kN/m 2 abs and a maximum fluid temperature of 450 K.

The water is circulated by either of two gear pumps connected in parallel. The flow passes through the coiled stainless-steel electrical preheater to the test-section inlet plenum. From the exit plenum the flow passes through a 5.1-centimeter-diameter pipe to a spray condenser. The coolant is supplied to the condenser by a centrifugal pump having a nominal capacity of $\sim 6.3 \times 10^{-3}$ cubic meter per second. From the condenser the flow passes into a multiple-tube heat exchanger cooled by cooling-tower water. In most cases the condenser is shut off, and the condensing is done in the heat exchanger.

The power to the test section and preheater is supplied by 70- and 250-kilowatt ac transformers, each regulated by a saturable core reactor. The two power supplies are interchangeable; either one can be connected to the test section or preheater, depending on the heat loads for a particular test.

Test Sections

The test sections used in this investigation are constructed of 0.584- and 1.219-centimeter inside diameter by 0.25-millimeter wall Inconel X tubing and are from 14.6 to 121.9 centimeters long. This tubing is specially rolled to limit the wall-thickness deviation to less than 3 percent of the nominal wall thickness. The test-section insidesurface roughness is measured by a surface analyzer before and after use for a number of the test sections. The maximum surface roughness (peak to valley) is ~5 nanometers. Several samples of tubing are used to measure the electrical resistivity of the Inconel X as a function of temperature. The results agree well with reference 15; the electrical resistivity of Inconel X may be considered independent of temperature for the range of this investigation.

Figure 2 shows a schematic diagram of a typical test section. The ratio of unheated length to inside diameter is 0.8 for test section 480-100M (table I) and 15 for all other test sections. (The test-section numbers give the inside diameter in thousandths of an inch followed by the ratio of heated length to inside diameter; the letter indicates the order of testing.) Power leads are connected to copper flanges that are silver-soldered to the tube. High temperature solder is used, and care is taken to ensure good contact and to avoid fillets in the heated portion of the test section. The end flanges are bolted

to the inlet and exit plenums and electrically insulated by Teflon and asbestos gaskets and seals. The exit plenum is fixed to the structural frame, while the inlet plenum is allowed to slide longitudinally to provide for thermal expansion. The test-section assembly is enclosed in a transparent shield to minimize external convective currents and to provide protection in case of failure.

Instrumentation

Fluid temperatures. - Plenum chambers (fig. 3) are provided at the test-section inlet and exit in order to obtain (at least in the all-liquid case) true bulk-temperature readings. The inlet- and exit-plenum temperatures, as well as other fluid temperatures around the loop are measured by copper-constantan thermocouples and recorded by a multipoint self-balancing potentiometer. This instrument is periodically calibrated with a standard potentiometer. The error in the bulk temperatures is estimated to be ± 0.3 K.

<u>Pressures</u>. - For most of the data reported herein, the fluid pressures at the inlet and exit plenums are measured with strain-gage-type transducers having a range of 0 to 690 kN/m^2 and a stated accuracy of 1/4 percent of full scale. The signal is recorded on a multichannel oscillograph. A 30-centimeter, 0- to 690-kN/m^2 Bourdon tube gage is connected to the exit plenum. In some cases, the test-section pressure drop is also measured with a 0- to 207-kN/m^2 differential gage connected across the pressure taps drilled through the copper flanges of the test section. Valves are provided to damp large amplitude gage pressure fluctuations or to shut off the gage completely. Under steady conditions, the correspondence between the transducers and gage readings is considered good.

<u>Flow rate</u>. - The flow rate is measured by one of three turbine-type flowmeters with overlapping ranges. All flowmeters are calibrated prior to their installation. The flowmeter signal is read from a frequency-converter indicator and on a digital frequency counter, both of which are calibrated by a signal generator.

<u>Power</u>. - Because a distorted ac output is obtained at low heating power levels (see ref. 1), a dynamometer-type wattmeter is used to measure the power input to the test section. The accuracy of this true rms instrument is 0.1 percent of full scale. A true rms, electronic-tube voltmeter is used to measure the test-section voltage drop and thus provide a check on the wattmeter.

Test-section wall temperatures. - The outside wall temperatures of the heated tube are measured by a number of 36-gage Chromel-Alumel thermocouples spot-welded in one longitudinal plane to the outside surface of the tube at various axial locations. The millivolt signal is recorded on a self-balancing potentiometer with a special filtering and isolation circuit to eliminate ac voltage pickup. Two 36-gage Chromel-Alumel thermo-

couples are spot-welded on the outside surface of the tube, both 0.635 centimeter from the exit end of the heated length, but circumferentially spaced 180° apart. These thermocouples are connected in parallel to yield a single average surface-temperature reading that is continuously recorded on a separate self-balancing potentiometer and serves as an overtemperature control.

PROCEDURE

Prior to filling, the system is flushed with deionized water, purged with nitrogen, and then evacuated. It is then filled with deionized, deaerated water. The water is circulated through the system and further degassed by boiling in the test section and venting to the atmosphere from the top of the condenser. Generally, water samples are taken and analyzed daily on a gas analyzer for gas content. The dissolved gas content is maintained at or below 3 parts per million (by weight).

The desired conditions for each run are established by adjusting the power to the preheater and test section and setting the pump speed and system pressure at selected values. When the mean inlet and exit bulk temperatures become constant with time, the data for that run are taken. Temperatures are automatically recorded on strip charts. Flow rate, power, and pressures are read from gages.

In order to check the wall thermocouples, runs are made in which no heat is applied to the test section, but the flow rate and preheater power are varied. Since the heat losses are small, the tube-outer-wall temperatures can then be checked against the water bulk temperatures. The wall-temperature recorders are adjusted so that the bulk and wall temperatures agree within the scatter of the wall-temperature measurement.

EXPERIMENTAL DATA

Tabulation

The experimental heat-transfer data are presented in tables I to VII. For each run, the mass velocity, heat flux, inlet and exit temperatures, the inner-wall-temperature profile, the exit quality, exit pressure, and, when available, the test-section pressure drop are listed. Each wall-temperature reading is classified as nonboiling, subcooled boiling, or net boiling.

Error Considerations

A maximum ± 1.7 K error in the inner-wall temperature is estimated in reference 1 for heat fluxes in excess of 3150 kW/m². This estimate is based on the maximum deviation in wall thickness, the accuracy of the outer-wall-temperature and heat-flux measurements, and the validity of the assumptions used in deriving the equation (1) for temperature drop through the heated wall. At the lower heat fluxes employed in the heat-transfer studies of this report, the errors should be less, perhaps in the vicinity of ± 0.5 K. Therefore, the wall temperatures are tabulated to the nearest 0.5 K. The error in the water temperature measurements is estimated to be ± 0.3 K (ref. 1); therefore, inlet and exit temperatures are tabulated to the nearest 0.1 K.

To estimate the heat loss from the test section, the heat-transfer coefficient to the air is estimated from reference 16, and the convective heat loss from the test-section surface is estimated to be less than 1 percent. Axial wall conduction calculations (ref. 17) based on measured temperature profiles indicate a maximum heat-transfer coefficient error of about 3 percent at the first measuring station, with error decreasing as length increases. During boiling, the rate of change of wall temperature with distance is usually quite small; thus, axial conduction is slight, and the assumptions used in deriving equation (1) for temperature drop through the wall are more valid.

Data Reduction and Terminology

Figure 4 shows typical temperature and voidage profiles for the case of subcooling at the inlet and net quality at the exit and the pertinent terminology. The point of boiling initiation, as determined from the wall-temperature profile, is at z_0 ; other parameters evaluated at this position are denoted by the subscript o. The subscript d denotes the point at which bubbles first detach from the tube wall. The point where the heat-balance quality is zero is indicated by the subscript 1. Any quantity which is corrected for non-equilibrium is indicated by a superscript prime. (Symbols are defined in the appendix.)

The heat input is determined from the wattmeter. The comparison between the wattmeter reading and the power computed from the voltage measurement and the known test-section resistance agree well. The heat flux is computed by dividing the wattmeter power by the tube inside area over the length $\, L_h^{}$. The outside wall temperature is obtained from the potentiometer millivolt signal by means of the calibrated temperature against emf curve for Chromel-Alumel thermocouples.

The inside wall temperature is computed as in references 1 and 17 by

$$0.593 (T_{wo} - T_{w}) + 0.000638 (T_{wo}^{2} - T_{w}^{2}) = qD \left[\frac{\ln\left(\frac{D_{wo}}{D}\right)}{1 - \left(\frac{D}{D_{wo}}\right)^{2}} - \frac{1}{2} \right]$$
(1)

which is derived for the following conditions: no heat loss from the outer surface, no axial heat flow within the wall, constant electrical resistivity, and a linear variation of thermal conductivity with temperature. Where the pressure drop is not determined experimentally, the pressure profiles are estimated using the correlations of references 18 and 2 for the subcooled- and net-boiling regimes, respectively. The liquid bulk temperature or vapor quality is then calculated by the following heat-balance equations, respectively:

$$T_{\ell} = T_{i} + \frac{4qz}{DGc_{p}}$$
 (2)

$$x = \frac{4qz}{DG\lambda} - \frac{c_p}{\lambda} (T_s - T_i)$$
 (3)

In comparing with net-boiling heat-transfer correlations, the fully developed all-liquid heat-transfer coefficient is required. The quantity $(h_\ell D/k)(DG/\mu_\ell)^{-0.8}$ is given graphically as a function of Prandtl number $c_p \mu_\ell/k$ in reference 17. Over the current range of interest, the relation for h_ℓ may be approximated by

$$h_{\ell} = 0.0232 \left(\frac{k}{D}\right) \left(\frac{DG}{\mu_{\ell}}\right)^{0.8} \left(\frac{c_{p}\mu_{\ell}}{k}\right)^{0.467}$$
(4)

RESULTS AND DISCUSSION

Typical Data

Large subcooling at inlet. - Typical heat-transfer data with large subcooling (~83 K) at the inlet are shown in figure 5. Local inner-wall and bulk temperatures are plotted against the axial distance from the start of heating in figure 5(a). The bulk temperatures

are based on an equilibrium heat balance. The resulting local heat-transfer coefficients, based on bulk temperature, are plotted against distance in figure 5(b). Papell's correlation of subcooled-boiling heat transfer (ref. 9) is shown for comparison. Three runs are shown with variable heat flux and constant mass velocity, exit pressure, and inlet temperature.

At the lowest heat flux, the inner-wall temperature does not rise above saturation temperature until near the exit, and no boiling occurs. The local heat-transfer coefficient decreases with distance in the thermal entrance region and then increases slightly due to the effect of the increasing temperature on physical properties. For the intermediate heat flux, the inner-wall temperature rises with distance rapidly enough that subcooled boiling is initiated in the thermal entrance region; the heat-transfer coefficient then increases with distance throughout the rest of the tube, all in the subcooled-boiling regime. Qualitatively similar results are seen for the highest heat flux although boiling is initiated nearer the inlet, net vapor is produced (exit quality $x_{\alpha} = 0.044$), and the boiling heat-transfer coefficients are higher. It should be noted that, for both runs with boiling in figure 5(a), the inner-wall temperature increases with distance in the subcooled-boiling regime; this effect is not predicted by most subcooled-boiling heattransfer correlations (e.g., refs. 6 to 8) but is of small magnitude. Throughout most of the subcooled-boiling regime, the experimental heat-transfer coefficients do not deviate greatly from Papell's correlation (ref. 9). But, the experimental data show less effect of distance (or subcooling) than predicted, and the deviation becomes large as the heatbalance quality approaches zero.

Small subcooling at inlet. - Typical heat-transfer data with small subcooling (~22 K) at the inlet are shown in figure 6; local inner-wall and bulk temperatures are plotted against axial distance from the start of heating in figure 6(a), and the resulting local heat-transfer coefficients, based on bulk temperature, are plotted against distance in figure 6(b). Papell's correlation of subcooled-boiling heat transfer (ref. 9) is again shown in figure 6(b) for comparison. Four runs are shown with variable heat flux and constant mass velocity and inlet temperature; the resulting exit quality ranges up to 0.247. The exit pressure is 119 kN/m^2 abs at the lowest heat flux and increases by approximately 7 kN/m^2 for each successive increase in heat flux; this increase in exit pressure is due to the increasing pressure drop between the boiler and condenser with increasing quality.

At the lowest heat flux, subcooled boiling is initiated somewhere near the middle of the tube, but the exact point is difficult to determine; the bulk condition at the exit is slightly subcooled. For the higher heat fluxes, subcooled boiling is initiated in the thermal entrance region, and net exit quality is produced. For all four runs, except very near the inlet, there is a general trend for inner-wall temperature to increase slightly with distance in the subcooled-boiling regime. For the three higher heat fluxes, at a

given position, h increases as q increases in the subcooled-boiling regime (fig. 6(b)). But then at a fairly low net quality, h takes on values of $\sim 57 \text{ kW/(m}^2)(\text{K})$ and does not vary much with heat flux, distance, or quality. The agreement with Papell's correlation (ref. 9) is poorer here than for figure 5, especially for the lowest heat flux, indicating that the range of applicability of the correlation is limited to subcoolings on the order of 20 K or greater.

Subcooled-Boiling Curves: Heat Flux Against Temperature Difference

A common way of presenting boiling heat-transfer data is to plot heat flux against wall superheat T_w - T_s . Figure 7 gives such a plot for an exit pressure of $\sim\!114~\rm kN/m^2$ abs; the data of this report and reference 1 as well as some limited data from reference 17 are shown. Two tube diameters and four length-to-diameter ratios are included. The wall-temperature measuring station 0.635-centimeter upstream of the test-section exit is used, except for the 1.219-centimeter inside diameter data of reference 17 where the station 1.905 centimeters upstream of the exit is used. Local subcooling T_s - T_ℓ ranges from 3 to 66 K. The range of mass velocity is from 0.67 to 141 kg/(sec)(m²), while the heat-flux range is from 101 to 11 400 kW/m². The correlations of Rohsenow (ref. 6), Forster and Greif (ref. 7), and Forster and Zuber (ref. 8) are shown for comparison. Although the data fall into a rather broad band (~22 K) and cannot be said to agree with any correlation, the figure is useful in summarizing the range of results obtained. It may be that the effect of other variables, such as mass velocity and subcooling must be accounted for. The correlation of reference 8 gives approximately the lower limit of q for a given T_w - T_s . But it appears that the correlation of either reference 6 or 7 would give reasonable approximate results for many applications.

Comparison With Net-Boiling Heat-Transfer Correlations

Most correlations of net-boiling heat-transfer coefficients relate $\,h/h_{\ell}\,$ to the Martinelli parameter

$$\boldsymbol{X}_{tt} = \left(\frac{1-\boldsymbol{x}}{\boldsymbol{x}}\right)^{0.9} \begin{pmatrix} \boldsymbol{\rho}_g \\ \boldsymbol{\rho}_{\boldsymbol{\ell}} \end{pmatrix}^{0.5} \begin{pmatrix} \boldsymbol{\mu}_g \\ \boldsymbol{\mu}_{\boldsymbol{\ell}} \end{pmatrix}^{0.1}$$

where h_{ℓ} is the heat-transfer coefficient for all-liquid flow at saturation temperature. Figure 8 gives plots of h/h_{ℓ} against x/(1-x) for mass velocities of ~ 0.68 and ~ 5.9

kg/(sec)(m²) at an exit pressure of ~117 kN/m² abs, from test section 480-100M (table I). Although some orderliness with regard to heat flux can be seen in both figures, it is apparent that the effect of mass velocity is not properly accounted for. The correlation of Dengler and Addoms (ref. 10) is shown for comparison. Although the data are not in agreement with the correlation, the slope of h/h_{ℓ} against x/(1-x) appears to agree for relatively high qualities in figure 8(a).

Similar plots are shown in figure 9 for a mass velocity of $\sim 0.68 \, \mathrm{kg/(sec)(m^2)}$ at exit pressures of ~ 114 and $\sim 26 \, \mathrm{kN/m^2}$ abs, from test section 480-50M (table II). The data of figures 8(a) and 9(a) show good agreement between the two test sections. But the experimental data show that, for constant heat flux, mass velocity, and quality, $\mathrm{h/h_{\ell}}$ (and also h) increases as pressure increases, whereas the correlation of reference 10 (and also those of refs. 11 to 14) predicts the opposite, due to the physical properties in $\mathrm{X_{tt}}$. Although the correlations of references 10 and 14 include a nucleate-boiling term, which is not directly a function of $\mathrm{X_{tt}}$, the contribution of the nucleate term is not of sufficient magnitude to reverse the pressure effect in either case. Thus these correlations in terms of $\mathrm{X_{tt}}$ are inadequate.

Definition of Heat-Transfer Coefficient During Boiling

In net boiling (equilibrium heat-balance quality x>0) the heat-transfer coefficient is usually defined as $q/(T_w-T_s)$, while in the subcooled-boiling regime, either $q/(T_w-T_s)$ or $q/(T_w-T_\ell)$ is commonly used. In figures 5(b) and 6(b) the latter definition, $h=q/(T_w-T_\ell)$, is used with $T_\ell=T_s$ for x>0. These heat-transfer coefficients are plotted against local equilibrium heat-balance quality x in figure 10(a). It can be seen that h increases rapidly with x in the subcooled region (x<0) and passes through a maximum in the low-quality range. In figure 10(b), $h_s=q/(T_w-T_s)$ is plotted against x for these same runs. It is difficult to find any consistent trends. These two definitions of the heat-transfer coefficient are not the only ones possible, however. Since some of the heat added to a fluid in subcooled boiling goes into the production of vapor, the remaining liquid has a lower average temperature T_ℓ than that calculated from an equilibrium heat balance. Performing a heat balance from the point of boiling initiation (subscript o) to any downstream point,

$$T_{\ell}^{\prime} = T_{\ell, o} + \frac{4q(z - z_{o})}{DGc_{p}} \left(\frac{1 - \overline{\gamma}}{1 - x^{\prime}} \right)$$
 (5)

where x^{ι} is the actual quality and $\overline{\gamma}$ is the net fraction of the heat added from $\,z_{_{\scriptstyle O}}\,$ to $\,z$

going into vaporization $x'/(x - x_0)$. There are several means available for predicting x' and $\overline{\gamma}$; for the present discussion, the following simple relation suggested by Levy (ref. 19) is used:

$$x' = x - x_d^{(x/x_d)-1}$$
 (6)

where x_d is the equilibrium heat-balance quality at the point where bubbles first depart from the surface; hereinafter, x_d is taken to be x_0 . The heat-transfer coefficient corrected for nonequilibrium, $h' = q/(T_W - T_U')$ is plotted against the equilibrium heat-balance quality x in figure 10(c). It can be seen that the range of h' values is much less than that of h or h_s , and the variation of h' with x is less than that of h or h_s .

Another reasonable plot to make is h' against x'; this is shown in figure 11. In both figures 10(c) and 11 there is an apparent effect of inlet temperature; other relations, suggested by Kroeger and Zuber (ref. 20), give similar results. This may be due in part to the contribution of nucleate boiling; but neither of the correlations (that of ref. 10 or 14), using nonequilibrium quality, agree well with the data. However, it should be noted that equation (6) is an assumed equation, used previously only for smaller subcoolings, and the assumption that $\mathbf{x}_0 = \mathbf{x}_d$ may contribute to the apparent subcooling effect. It appears that \mathbf{x}' is calculated to be too large; pressure drop calculations also indicate that \mathbf{x}' is too large. Therefore, although it might be possible to obtain \mathbf{h}' as a function of \mathbf{x} or \mathbf{x}' , \mathbf{q} , \mathbf{G} , and physical property values for a given initial subcooling, it appears that a nonequilibrium model valid over a wide range of subcooling and pressure is still required before a general correlation of the data will be successful.

SUMMARY OF RESULTS

The major results of this investigation of heat transfer to low-pressure water in subcooled and net-quality boiling in electrically heated tubes may be summarized as follows:

- 1. Axial inner-wall temperature distributions are tabulated for subcooled and net boiling in 0.584- and 1.219-centimeter inside diameter tubes over the following ranges of test variables: mass velocity from 0.67 to 141 kg/(sec)(m 2), heat flux from 43.8 to 11 400 kW/m 2 , exit pressure from 24 to 690 kN/m 2 abs, qualities up to 0.65, and liquid subcoolings as high as 151 K. As no satisfactory general correlations are available, these experimental data over a wide range of test conditions should be useful to the designer.
- 2. The subcooled-boiling data show only approximate agreement with the correlations of Rohsenow, Forster and Greif, Forster and Zuber, and Papell. In some cases the

tube-inner-wall temperature increases with increasing distance from the start of heating in the subcooled-boiling regime, which is not predicted by any existing correlation; however, this effect is small.

- 3. Plots of net-boiling heat-transfer coefficients against heat-balance quality do not agree with any existing correlation; the data show that, for constant heat flux, mass velocity, and quality, the local heat-transfer coefficient increases as pressure increases whereas the correlations predict the opposite.
- 4. Heat-transfer coefficients, based on a wall-to-liquid temperature difference corrected for nonequilibrium, vary less with quality and show more consistent trends than do heat-transfer coefficients based on either wall-to-bulk or wall-to-saturation temperature differences. However, a completely satisfactory correlation of the data is not attained because no presently available nonequilibrium model appears to be valid over the entire range of liquid subcoolings of this study.

Lewis Research Center,

National Aeronautics and Space Administration, Cleveland, Ohio, April 1, 1971, 120-27.

APPENDIX - SYMBOLS

- c_{p} liquid heat capacity, J/(kg)(K)
- D tube inside diameter, m
- D_{wo} tube outside diameter, m
- G mass velocity, kg/(sec)(m²)
- h heat-transfer coefficient, $q/(T_w T_{\ell})$, $W/(m^2)(K)$
- h_{ℓ} calculated all-liquid heat-transfer coefficient (eq. (4)), $W/(m^2)(K)$
- h_S boiling heat-transfer coefficient, $q/(T_W^2 T_S^2)$, $W/(m^2)(K)$
- k liquid thermal conductivity, W/(m)(K)
- L_h heated length of test section, m
- L₁₁ unheated length of test section, m
- P pressure, N/m² abs
- ΔP test-section pressure drop, N/m^2
- q heat flux, W/m^2
- T temperature, K
- X_{tt} Martinelli parameter, $\left(\frac{1-x}{x}\right)^{0.9} \left(\frac{\rho_g}{\rho_{\ell}}\right)^{0.5} \left(\frac{\mu_g}{\mu_{\ell}}\right)^{0.1}$, dimensionless
- x vapor quality (eq. (3)), dimensionless
- z axial distance from start of heating, m
- $\overline{\gamma}$ fraction of heat added going into vaporization, $x'/(x x_0)$, dimensionless
- λ enthalpy of vaporization, J/kg
- μ viscosity, kg/(sec)(m)
- ρ density, kg/m³

Subscripts:

- d point where bubbles first detach from tube wall
- e test-section exit
- g gas property
- i test-section inlet
- liquid property, or based on liquid bulk temperature

- o onset of boiling, as determined from wall-temperature profile
- s saturation, or based on saturation temperature
- w tube inner wall
- wo tube outer wall
- 1 point where heat-balance quality is zero

Superscript:

value corrected for nonequilibrium

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TABLE I. - HEAT-TRANSFER DATA FOR 1. 219-CENTIMETER INSIDE DIAMETER BY 121.9-CENTIMETER-LONG TEST SECTION 480-100 M [G = mass velocity, kg/(sec)(m²); q = heat flux, kW/m²; T_i = inlet temperature, K; T_e = exit temperature, K; P_e = exit pressure, kN/m² abs; x_e = exit quality.]

Run	G	q	T _i	Т _е	Pe	x _e							A	xial dis	ance fro	m start	of heatin	g, z, cn	n						
							0.635	1.27	2.54	5.08	10.16	15.24	22.86	3C.48	38.10	45.72	53.34	60.95	68.58	76.20	86.36	96.52	106.7	116.8	121.3
														Inn	er-wall t	emperat	ure, T _w	, к							
30 31	5.93 5.91 5.95 5.91 5.88	167 166	29C.1 311.5 325.4 344.6 357.6	337.6 352.1 370.4	703 703 703		a310.0 a326.0 a339.0 a354.0 a365.0	a 344.0 a 360.0	a348.0 a363.0	a340.0 a351.5 a366.5	a345.0 a354.0 a370.0	^a 347.0 ^a 358.0 ^a 372.0	a350.0 a359.0 a375.0	a351.0 a361.5 a377.0	a353.0 a363.0 a379.0	a356.5 a365.0 a381.5	a357.0 a366.5 a383.0	^a 358.0 ^a 369.0 ^a 385.0	a360.0 a370.5 a387.0	^a 361.5 ^a 373.0 ^a 389.0	a364.0 a375.0 a392.0	a366.5 a378.0 a396.0	a368.0 a381.0 a398.0	a371.0 a383.0 a401.0	a374.0 a386.5 a403.0
34 46 47 66 67	5.95 5.91 5.93 5.95	287 328 413 545 580	292.1 295.1 295.1 294.8 295.4	337.3 .346.5 360.1 375.4 375.4	116 117 117 122	.008	a 321.0 a 325.0 a 337.0 a 3373.0 a 373.0	a335.0 a341.5 a355.0 a376.5 a378.0	a342.0 a351.0 a364.0 a388.0	a351.0 a358.0 a374.0 a390.0 b388.0	a362.0 a373.0 a380.0 b391.0 b388.0	8362.0 8375.0 8378.0 8391.0	a366.5 a374.0 b382.0 b394.0 b390.0	a371.5 a369.0 b382.0 b394.0	a369.0 a371.5 b384.0 b391.5	a369.0 a374.0 b385.0 b355.0 b395.0	a369.0 a385.0 b385.0 b393.0	a372.0 a381.0 b386.0 b391.0 b395.0	a374.5 a384.5 b386.0 b391.5 b390.5	a378.0 b386.5 b388.0 b396.5	a378.0 b386.5 b389.0 b394.0	8381.5 b386.5 b391.0 b396.5	a388.0 b390.0 b393.0 b397.0	8388.0 8392.0 8396.0 8399.0	8392.0 8394.0 8394.0 6402.0
36 37 38 39 48	5.93 5.95 5.95	206 240 300	318.2 318.2	350.7 357.1 366.5	116 116		a331.0 a334.0 a339.0 a344.0 a363.0	^a 341.0 ^a 347.0 ^a 353.0	a346.5 a351.0 a361.0	a351.5 a359.0 a368.0	a355.0 a364.0 a375.0	a359.0 a367.0 a379.0	a361.5 a370.0	18364.0 18372.0	18365.0 8374.0	a _{37.8.0} a _{37.7.0}	a378.0	a371.5 a380.0 b386.5	a374.5 a383.0 b386.5	2376.0 2385.0	8377.0 9385.0	8380.0 5384.0	b383.0	b383.0	b384.0
41 42 43 44 45 69 70 71 72 73 74	5.95 5.883 5.893 5.55 5.95 5.95 5.95 5.95 5.95 5.95 5.	125 164 206 237 284 333 418 473 556 719 783	359.8 359.8 360.9 360.1 361.8 362.1 363.2 363.4 363.4	375.4 374.3 374.3 375.4 375.7 375.6 376.6 376.6	122 122 d122 d124 d125 d124 d126 d130 d133	.005 .013 .028 .036 .052 .065 .070 .093 .111 .135 .161	a365.0 a367.0 a367.0 a367.0 a375.0 a375.0 a375.0 b387.0 b387.0 b396.5 b396.5 b396.5 b396.5	2372.0 2372.0 2376.0 2376.0 2376.0 2390.0	a375.5 a385.0 a383.0 b390.0 b388.5 b394.0 b393.5 b397.0 b396.5 b397.0	8378.05 8378.05 8388.00 8384.00 8384.00 8395.00 8395.00 8396.00 8396.00 8396.00	8388.00 8387.00 8387.00 8387.00 8386.00 8394.00 8394.00 8394.00 8397.00 8307.00 8307.00 8307.00 8307.00 8307.00 8307.00 8307.00 8307.00 830	5382.5 5382.5 5387.0 5386.0 5386.0 5391.0 5392.5 5394.0 5394.0 5396.0 5396.0	2386.0 2386.0 2387.0 2387.0 2387.0 2387.0 2387.0 2387.0 2397.0 2398.0 2398.0 2398.0 2398.0 2398.0 2398.0 2398.0	2388.0 2388.0 2387.0 2387.0 2387.0 2390.0 2390.0 2390.0 2390.0 2390.0	5384.0 5387.0 5387.0 5386.5 5386.5 5392.0 5394.5 6394.5 6397.0 6399.0 6399.0	03889.0 03887.0 0387.5 0387.5 0392.0 0392.0 0397.5 0400.0 0397.5	0393.0 0385.5 0386.0 0386.0 0393.0 0393.0 0395.5 0398.0 0400.5	5389.0 5389.0 5386.5 6386.0 6393.0 6393.0 6394.0 6397.0 6399.0 6400.5	0387.0 0389.0 0386.0 0387.0 0389.0 0393.0 0394.0 0394.0 0399.0 0401.0	0389.0 0386.0 0386.5 0387.0 0388.5 0391.0 0393.0 0393.0 0393.0 0393.0	0389.0 0388.0 0385.9 0386.0 0388.9 0390.0 0393.0 0393.0 0395.0 0399.5 0401.0	0387.0 0387.0 0384.0 0384.0 0386.0 0388.0 0391.0 0391.0 0395.5	0389.0 0383.0 0383.0 0383.5 0385.0 0387.5 0389.0 0390.0 0393.0	C388.0 C382.0 C382.0 C382.5 C382.5 C384.0 C387.0 C387.0 C391.5 C395.5	2388.0 2388.0 2384.0 2383.0 2384.0 2385.5 2387.0 2388.0 2388.0 2390.0 2390.0 2393.0

51 52 53	2.39 2.35 2.34 2.34	134 163 199 231 284	358.2 355.9 357.6 356.5	374.8 374.3 373.7 374.3	1?3 d122 d123 d123	.062 .079 .112 .135	a379.0 a370.5 a385.0 a383.0 a382.5	2386.0 2388.0 2385.0 2386.5	539C.0 5386.0 5383.0 5384.0	5384.5 5382.5 5384.0 5384.5	0387.0 0385.0 0386.0 0385.5	0384.0 0384.0 0384.0 0384.0	5384.0 5384.0 5384.0	1367.0 1365.0 1386.0 1386.0	C386.0 C384.0 C384.0	C386.0 C385.0 C385.0	C386.0 C384.0 (c) C386.0	C386.0 C386.0 C386.0	C387. C386. C387. C387.	5 c383.0 0 c387.0 0 c385.5 0 c386.0 0 c386.0 5 c386.5 5 c383.0 0 c383.5	C387.0 C385.0 C386.0 C385.0	C386.0 C384.0 C385.0 C384.0	C384.0 C384.0 C384.0	C385.0 C383.5 C383.5	C384.0 C383.0 C383.0 C383.0
61 62 53	0.68 0.69 0.67 0.69	63 79 103 127 155 198 234	354.8 354.3 354.3 357.1 355.4	375.4 375.1 375.1 375.1 374.2 375.1	123 123 d123 d123 d123 d123 d123	.118 .158 .218 .295 .357 .491	a37,.0 a380.0 a386.0 b386.0 c385.0 c381.0 b381.5	5381.5 5388.0 5391.0 5384.0 5385.0 5385.0	0389.0 0383.0 0382.5 0382.5 0382.0	5386.5 5384.0 5384.0 5383.0 5381.3 5379.0	0384.0 0384.0 0385.0 0383.0 0382.0	0384.0 0383.0 0382.0 0382.0 0382.0	0382.0 0380.0 0383.0 0384.0 0385.0	C381.0 C383.C C384.5 C385.0 C386.0	°381.0 °382.0 °385.0 °386.0 °386.0 °386.0	C381.0 C3 2.0 C385.0 C386.0 C386.0	C381.0 C382.0 C384.5 C386.0 C386.0	C381.0 C385.5 C387.0 C387.0 C387.0	C382 - C383 - C385 - C387 - C387 - C387 -	5 C380.0 0 C382.0 0 C383.0 0 C385.0 0 C387.0 0 C387.0 0 C387.0 0 C388.0 (c)	C382.0 C383.0 C386.0 C386.0 C386.0 C386.0	C382.0 C383.0 C385.0 C386.0 C386.0 C386.0 C386.0	C382.0 C383.0 C385.0 C386.0 C386.0 C345.0	C383.0 C384.0 C385.0 C386.0 C386.0	C383.0 C383.0 C384.5 C385.0 C385.0 C385.0 C385.0
2 3 4	1.34 1.00 0.69	100	298.7 293.7 294.3	370.4	99	.032	a334.5	a343.0	a354.0	2368.0	D378.0	D374.0	D376.5	D380.C	D381.C	D384.0	D381.0	b377.5	b381 .	O b380.0 O b382.0 O c385.0	D380-0	b380-5	(c)	b382.0 c383.0 c390.0	C379.0
5 6 7 8 9	1.00 1.34 1.00	131 132 132	293.2 292.6 293.2	378.7 373.7 375.9	d ₁₁₀	.074 .020	a345.0 a341.0 a346.0	a _{356.0} a _{351.0} a _{358.5}	^a 369.0 ^a 364.5 ^a 371.0	b383.0 a374.0 b385.0	³ 382.0 ³ 382.0 ³ 380.0	b384.5 b384.0 b381.0	b384.5 b378.0 b381.0	b383.0 b383.0 b381.0	b387.0 b384.0 b383.0	b349.0 b382.0	b387.5 b38215 b385.5	b383.0 b385.5	b387. b383. b384.	0.5384.0 0.5385.0 5.5385.0 0.5384.0 0.6386.0	0386.0 0385.0	C387.5 D383.0 C386.0	(c) (b) (c)	C385.0 C390.0 C387.0 C389.0	°388.0 °383.0 °383.0
	1.34	192 190	295.9 296.5 296.5 296.5	374.3 374.3	117	.106	a ₃₆₈ .0	a375.0	b389.0	b383.0	b384.0 b385.5	5386.0 5387.0	D385.0	b388.0	5388.0 5389.0	b389.0	D387.0	b387.0	b386 .	5 b388.0 0 c387.0 0 c387.5 5 c388.0	C387.0	C388.0	(c)	b390.0 c391.0 c392.0 c392.0	C389.0
18	2.37	251	295.4 297.6 296.5	374.3	118	.041	a362.0	a378.0	b383.0	b382.0	b384.0	b385.0	b385.0	b387.0	b388.0	D385.0	b389.0	b389.0	b389.	5 0381.0 0 0389.0 5 0387.5	Dagg_n	C389.0	1 751	b391.0 c392.0 c392.0	C390.0
^b Sube ^c Net- ^d Pre		boiling y boili oscilla				own.				'				•							L				

TABLE II. - HEAT-TRANSFER DATA FOR 1. 219-CENTIMETER INSIDE DIAMETER BY 61-CENTIMETER-LONG TEST SECTION 480-50 M

[G = mass velocity, kg/(sec)(m²); q = heat flux, kW/m²; T_i = inlet temperature, K; T_e = exit temperature, K; P_e = exit pressure, kN/m² abs; x_e = exit quality.]

Run	G	q	T _i	Т _е	P _e	x _e					A	xial dist	ance fro	m start	of heatir	ıg, z, cr	n				
}					}		0.635	1.270	2.540	5.080	10.16	15.24	20.32	25.40	30.49	35.56	40.64	45.72	50.80	55.88	60.32
	į į				Ì							Inne	er-wall t	emperat	ure, T _w	, к					
17 18	5.89 5.91	253 315	294.8 294.8	314.8 320.9	110		a335.0 a345.0	a343.0 a354.0	⁸ 354.0 ⁸ 363.0	a355.0 a370.0	a365.0 a375.0	a370.0 a376.5	a372.0 a376.5	^a 375.0 ^a 378.0	a376.0 a379.0	a377.0 a379.0	a377.0 a379.0	a378.0 b380.0	a379.5 b379.0	b380.0 b379.0	b377.0 b380.0
19 20 21	5.92 5.92 5.89	258	323.7 324.0 324.0	338.7 344.6 350.4	110 110 110		^a 347.0 ^a 356.5 ^a 366.5	^a 353.0 ^a 363.0 ^a 373.0	^a 360.0 ^a 368.0 ^a 375.0	^a 361.0 ^a 375.0 ^b 378.0	a364.0 a376.5 b375.0	^a 368.0 ^a 376.5 ^b 378.0	^a 369.0 ^a 377.0 ^b 379.0	a370.0 b379.0 b378.0	a373.3 b379.3 b379.3	a374.0 b379.0 b380.0	^a 375.0 ^b 379.0 ^b 379.0	a376.0 b381.0 b378.0	^a 378.0 ^b 379.5 ^b 380.5	a378.0 b380.0 b386.5	a378.0 b379.0 b385.0
13 14 15 16	2.38 2.38 2.36 2.38	192 254	295.9 295.4 294.8 294.8	321.5 333.2 345.9 358.7	107 110 110 110		1ª367.0	a344.0 a363.0 b380.0 b380.0	a353.0 b376.0 b377.0 b380.0	a366.5 b376.0 b380.0 b380.0	b374.0 b374.0 b380.0 b380.0	b379.0 b376.0 b377.0 b384.0	b379.0 b376.0 b383.0 b384.0	b379.0 b374.0 b384.0 b384.0	b379.0 b376.5 b384.0 b385.0	b381.0 b375.0 b384.0 b385.0	b379.0 b375.0 b384.0 b385.0	b379.0 b377.0 b385.0 b386.0	b382.0 b378.0 b385.5 b386.0	b382.0 b377.0 b386.5 b385.0	b379.0 b376.5 b385.0 b385.0
22 23 24 25 26	2.38 2.38 2.38 2.38 2.38 2.38	128 192 258	320.9 320.9 320.7 320.4 320.4	340.9 345.9 358.7 371.5 374.3	110 110 110 110 110	.016	a347.0 a354.0 a369.0 b383.0 b385.0	a354.0 a361.5 a380.0 b381.0 b385.0	a361.0 a370.0 b381.0 b375.0 b385.0	8367.0 b376.5 b381.5 b375.0 b385.0	a373.0 b374.0 b381.5 b379.0 b384.0	b381.5 b375.0 b383.0 b382.0 b386.5	b374.0 b377.0 b383.0 b382.0 b385.0	b375.0 b375.0 b384.0 b385.0 b385.0	b374.0 b378.0 b384.0 b385.0 b385.0	b376.0 b378.0 b384.0 b385.0 b385.0	b376.5 b378.0 b384.0 b385.0 b385.0	b376.5 b378.0 b384.0 b386.0 b385.0	b378.0 b378.0 b384.5 b386.0 b385.5	b375.0 b378.0 b384.0 b386.0 c385.0	b374.0 b375.0 b386.5 b384.0 c385.0
37 38 39	2.36	102 119 198	344.3 345.4 345.4	368.7	98 98 108	-019	a363.0 a369.0 b385.0	a370.0 a376.0 b384.0	^a 376.5 ^a 378.0 ^b 382.0	b381.0 b379.0 b384.0	b379.0 b378.0 b381.0	b381.0 b380.0 b384.0	b380.0 b380.0 b383.0	b380.0 b380.0 b383.0	b380.0 b380.0 b383.0	b380.0 b380.0 b383.0	b380.0 b380.0 b383.0	^b 380.0 ^b 379.0 ^b 384.0	b380.0 b380.0 c384.0	b380.0 b380.0 c384.0	b378.0 b379.0 c384.0
6 7 8 9 10 11 12	0.69 0.69 0.68 0.69	101 124 154 191	297.1 297.1 297.1 298.7 298.2	375.4	109	.014 .055 .105 .191 .262	a327.0 a346.5 a355.0 a368.0 b378.0 b384.0 b384.0	a333.0 a358.0 a356.5 b381.5 b378.0 b384.0	a342.0 a370.0 a379.0 b381.5 b379.0 b383.0 b383.0	a350.0 a379.0 b379.0 b381.0 b380.0 b384.0	a360.0 b379.0 b381.0 b380.0 b380.0 b382.0 b382.0	8366.5 9380.0 9382.0 9379.0 9384.0 9385.0 9384.0	a371.0 b381.5 b383.0 b383.0 b383.0 b383.0 b383.0	a371.5 b381.5 b383.0 b383.0 b383.0 b383.0 c383.0	a374.0 b383.0 b383.0 b383.0 b383.0 c384.0	358.0 b381.5 b383.0 b383.0 c384.0 c385.0	a361.0 b381.5 b381.5 b383.0 c384.0 c385.0	8367.0 9383.0 9383.0 9383.0 9385.0 9385.0	a374.5 b381.5 b381.5 c383.0 c385.5 c386.5	a376.5 b382.0 c382.0 c383.0 c385.0 c386.5	a376.5 b382.0 c383.0 c383.0 c384.0 c386.0
27 28 29 30	0.71 0.69 0.69 0.69		336.5 326.5 325.4 325.4	376.5	117	1 -234	10386. O	1 D385.0	I∨385.0	1 D3 86 - D	5383.0 5385.0 5383.0 5382.0	1 123 R S - N	 	1 C3 85 0	CARK. 1	I ∪appa ∩	U227 n	Capa n	CABBA	Canan	C384.0 C386.5 C388.0 C391.0
31 32 33 34			348.7 349.8 348.7 350.9	375.9 375.9	113	.111 .198 .283 .369	b386.0 b386.5 b386.0 b385.0	b384.0 b385.0 b386.0 b385.0	b384.0 b385.0 b383.0 b383.0	0384.0 0384.0 0383.0 0383.0	b382.0 b382.0 c381.5 c382.0	0384.0 0384.0 0385.0 0385.0	0383.0 0385.0 0385.0 0385.0	°385.0 °385.0 °385.0 °385.0	C385.0 C385.0 C386.0 C386.0	°385.0 °384.0 °°386.0 °389.0	c385.0 c384.0 c387.0 c389.0	C385.0 C388.0 C388.0 C389.0	C386 .5 C387 .0 C388 .0 C390 .5	C386.0 C388.0 C388.0 C390.0	385.0 387.0 388.0 390.0
41 42 43 44	0.69 0.69 0.69	126	295.9	338.4 338.2 1338.2 338.2	d ₂₅	.045 .082 .164 .241	(e)	(e) (e) (e) (g)	(e) (e) (e) (e) (g)	(e) (e) (e) (g)	(e) (e) b351.5 (g)	(e) (e) (e) (g)	(e) b351.5 c352.0 (g)	(e) b349.0 c354.0 (g)	b350.0 c348.0 c348.0 (g)	b349.0 c350.0 c350.0 (g)	^c 348.0 ^c 351.5 ^c 351.5 (g)	^c 348.0 ^e 353.0 ^c 353.0 (g)	^c 350,(^c 353,(^c 353,(_g)	°350.0 °353.0 °353.0 (g)	347.0 349.0 351.5 355.0
45 46 h47 48	0.69	127	335.4	338.7	d26	.129 .152 .234 .314	a348.0 b352.0 b353.0 b352.0	a349.0 b350.0 b353.0 b352.0	b351.0 b351.5 c354.0 c352.0	C350.0 C351.5 C356.0 C353.0	C351.0 C352.0 C359.0 C359.0	C350.0 C354.0 C359.0 C355.0	°352.0 °355.0 °358.0 °356.5	°354.0 °356.5 °360.0 °357.0	°355.0 °356.5 °358.0 °356.0	c355.0 c357.0 c360.0 c359.0	C354.0 C357.0 C350.0 C359.0	c354.0 c356.5 c359.0 c359.0	^C 354.0 ^C 356.5 ^C 359.0 ^C 358.0	354.0 355.0 358.0 356.5	0 ⁰ 352.0 0 ⁰ 352.0 0 ⁰ 353.0 5 ⁰ 352.0

^aNonboiling.

bSubcooled boiling.

CNet-quality boiling.

^dEstimated from exit temperature

eVery unsteady.

fTemperature oscillation.

gAutomatic shutdown did not allow measurement.

hAutomatic shutdown after data taken.

TABLE III. - HEAT-TRANSFER DATA FOR 1, 219-CENTIMETER INSIDE DIAMETER BY 61-CENTIMETER-LONG TEST SECTION 480-50

[G = mass velocity, kg (sec)(m²); q - heat flux, kW m²; T₁ - inlet temperature, K; T_e = exit temperature, K; P_e : exit pressure, kN/m² abs; x_e = exit quality; ΔP = pressure drop, kN 'm².]

Run	G	q	T ₁	T _e	Pe	x _e	ΔP					A	xıal dist	ance fro	m start	of heatin	g, z, en	n				
								2.540	7.645	12.70	17.78	22.89	27.94	33.02	38.10	43.18	48.25	53.34	58.42	59.05	59.72	60.35
													Inne	r-wall t	emperat	ure, T _w	, к					
4 5 8	33 5.93 5.93	545		323.7 334.3 333.2	115			(·)	(:)	(:)	(3)	400.0 255.0 406.0	403.0	403.6	463.0	64U3.6	b403.0	U404.J	b398.0	D389.0	D395.0	(b)
		226 517		374.8 375.4		.040	34 34	400.0 410.0	1401.5 1413.0	1401.0 1413.0	1400.0 1412.0	4399.0 409.0	399.0 406.0	-398.0 -1405.0	1397.0 1403.0	1396.0 1401.0	1395.0 1400.0	¹ 394.0 1399.0	d394.0 d398.0	d391.5 d395.0	d393.0 d397.0	d390.0 d395.0
12	12.0	924	294.8	322.1 332.1 343.7	117			405.0	411.5	410.0	410.0	399.0 409.0 413.0	410.0	409.0	405.0	408.0	C409.0	438.0	6401.0	396.5	D404.0	D400.0
20	10.4 10. 10. 	1359 1359 1343 1359 1352	385.4 384.3	374.8 377.1 379.8 378.7 377.1 377.6 377.1 375.9 374.8	210 187 173 170 163 154	.061 .078 .102 .163 .201 .241 .303 .377	33 55 45 42 41 36	418.0 428.0 421.0 419.0 419.0 416.5	398.0 430.0 426.0 423.0 422.0 421.5 420.0	423.0 430.0 419.0 422.0 421.0 420.0	1423.0 1430.0 1426.0 1423.0 1434.0 1421.5 1420.0	1414.0 1419.0 1428.0 1424.0 1421.5 1422.0 1420.0 (e)	419.0 428.0 425.0 422.0 423.0 421.5 420.0	418.0 427.0 426.0 421.5 422.0 421.0 420.0	415.0 426.0 422.0 420.0 421.0 420.0	1413.0 1424.0 1421.5 1419.0 1420.0 1419.0 1417.0	3412.0 3423.0 3420.0 3413.3 3418.0 3418.0 3415.3	410.0 421.0 418.0 416.5 416.5 416.5	d418.0 d415.0 d415.0 d414.0 d413.0	d411.0 d416.5 d414.0 d413.0 d412.0 d411.5	d407.0 d418.0 d415.0 d413.0 d413.0 d413.0 d413.0	d405.0 d415.0 d413.0 d410.0 d411.0 d411.0
28 29 30 31 32 33	.0. .43 4.20 3. 2	1589 1589 1589 1992	389.8 388.2 389.8	378.2 377.1 376.5 375.9 375.4 374.8	221 202 195 186	.149 .204 .315 .360 .424	77 72 70	432-0 431.0 430.0 425.0	435.0 3431.0 3431.5 3428.0	434.0 431.0 430.0 435.0	1436.5 1434.0 1435.0 1431.0	434.0 434.0 432.0 430.0 430.0	434.0 433.0 430.0 430.0	1435.0 1434.0 1435.0 1432.0	433.0 431.5 430.0 430.0	1432.0 1431.5 1430.0 1429.0	432.0 427.0 427.0 3427.0	430.0 428.0 427.0 427.0	d425.0 d424.0 d423.0 d421.5	d _{421.5} (d) (d) d _{419.0}	d425.0 d424.0 d422.0 d420.0	d421.5

a Nonboiling.

TABLE IV. - EXPERIMENTAL DATA FOR 0.584-CENTIMETER INSIDE DIAMETER BY 61-CENTIMETER-LONG TEST SECTION 230-100

[G = mass velocity, kg.'(sec)(m²); q - heat flux, kW m²; T_1 - inlet temperature, K; T_e = exit temperature, K; P_e = exit pressure, kN/m² abs; ΔP = pressure drop, kN/m²; x_e = exit quality.]

Run	G	q	T_{i}	T _e	\mathbf{P}_e	ΔP	$^{\mathrm{x}}e$					-	Axial dis	tance fr	om start	of heati	ng, z, cı	m				
								5.080	10.16	15.24	20.32	25.40	30.48	35.56	40.64	45.72	50.83	55.88	57.15	57.15	57.78	57.78
													Inn	er-wall	temperat	ture, T _w	, к					
		1790 2153		350.4 362.6												1403.0 1403.0						
72 73 74	46.2	3215 3467 3845	294.3 294.8 295.4	333.2 355.4 363.7 370.4 373.2	114 114 114	1		(.) 400.0 413.0	395.0 407.2 415.0	400.0 414.0 419.0	408.0 415.0 420.0	4405.5 416.0	409.0 413.0 415.0	-407.0 -414.0 -415.0	-403.0 -413.0 -414.0	389.0 -405.0 8409.0 4412.0 3414.0	4403.0 411.5 414.0	2473.0 2478.0 2479.0	(a) a408.0 a408.0	a402.0 a405.0 a408.0	a401.5 a405.0 a405.0	6402.0 6405.0 6407.0

a Subcooled boiling.

bSubcooled boiling.

c Fluctuating.

dNet-quality boiling.

eAutomatic shutdown.

bNonboiling,

Error

An error occurred while processing this page. See the system log for more details.

1	Axial dis	stance	from s	tart of h	eating,	z, cm	1.27	3.81	6.35	8.89	11.43	13.97	16.51	19.05	21.59	24.13	26.67	27.30	27.90	28.57
. 16 3L 66	27.8 2 28.1 2 27.8 1	244	310.9	334.3 346.5 365.4	115	13 13 24	b413.0	b414.0	b416.0	b419.0	b418.0	b413.0	b410.0	b412.0		b407.0	b436.5	b405.0	b410.0 b408.0 b410.0	b404.0
90 160 110	14.3 1 14.4 1 14.4 2	806	290.9	339.8 349.8 357.6	114	9 10 10	b405.0 b403.0 b403.0	b405.0 b407.0 b407.0	b406.0 b409.0 b408.0	b408.0 b411.0 b411.0	b409.0 b411.0 b410.0	b405.0 b407.0 b408.0	b408.0 b409.0 b410.0	b408.0 b410.0 b411.0	b405.0 b408.0 b410.0	b406.0 b408.0 b409.0	b403.0 b406.0 b405.0	b404.0 b404.0 b404.0	b411.0 b410.0 b411.0	b400.0 b402.0 b404.0
13L 14L 15L	14.1 2 14.3 2 14.3 2	348	292.1	359.3 368.2 380.9	310	8 9 10	a431.0 b430.0 b433.0	a435.0 b434.0 b437.0	a436.0 b434.0 b437.0	b438.0 b436.0 b438.0	b438.0 b435.0 b438.0	b438.0 b436.0 b438.0	^b 440.0 ^b 438.0 ^b 440.0	b438.0 b439.0 b439.0	b435.0 b435.0 b437.0	b438.0 b439.0 b439.0	b434.0 b435.0 b436.5	b430.0 b431.5 b433.0	b434.0 b435.0 b435.0	b430.0 b431.5 b433.0
176 196	14.1 2			388.2 395.4		14 14	^a 431.5 ^a 431.5	b434.0 b435.0	b434.0 b434.0	b435.0 b434.0	b435.0 b434.0	b435.0 b433.0	b436.0 b434.0	b436.5 b434.0	b433.0 b433.0	b436.0 b433.0	b433.0 b431.5	^b 429.0 ^b 428.0	b433.0 b431.5	b430.0 b429.0
216. 230.	∠7.3 3 ∠7.0 2			391.5 395.9		28 26	a435.0 a435.0	b438.0 b437.0	b437.0 b437.0	b439.0 b439.0	b439.0 b439.0	b438.0 b438.0	^b 440.0 ^b 439.0	b442.0 b440.0	b436.0 b435.0	^b 440.0 ^b 438.0	b436.0 b434.0	b432.0 b431.0	b436.0 b436.0	b434.0 b433.0
25L 26C	28.7 28.5					8 8	a444.0 a443.0	a446.5 b446.5	b448.0 b446.5	b449.0 b449.0	b449.0 b448.0	b450.0 b448.0	b451.5 b449.0	b449.0 b449.0	b449.0 b448.0	^b 449.0 ^b 449.0	b446.5 b446.5	b444.0 b444.0	b449.0 b446.5	b447.0 b445.0
2 86.	14.4 2	607	314.3	396.5	448	12	a441.0	D445.0	b445.0	b448.0	b446.5	b447.0	b448.0	b448.0	b446.0	b448.0	b445.0	b443.0	b445.0	b442.0
30C 37L 38L	70.1 6 70.2 6 70.2 6	493	299.3 299.8 300.4		117 117 117	26/ 30 35/ 46 52/ 70	a399-0	a409-0	a414.0	b418.0	b416.5	0416.5	b419-0	b423.0	b415.0	b421.0	b416.0	b412.0	b413.0 b414.0 b416.5	b414.0
416 416 426 436	70.2 4 70.5 5 70.1 5 70.1 6	169	325.4 326.5 325.9 326.5	357.6 360.9 363.2 367.6	117 117 117 117	63/ 66 71/ 74 80/ 83 96/101	5417.0 5419.0	C421.5	D419.0	b422.0	0420.0 0424.0	0421.0 0424.0	0423.0 0425.0	b425.0	b419.0	b422.0	b417.0	b415.0	b414.0 b415.0 b418.0 b421.0	b414.0
44L 45L	70.7 6 70.1 6		324.8 324.8	366.5 368.7	117 117	92/105	6423.0 6427.0	[≿] 429.0 [≿] 433.0	b428.0 b431.5	b430.0 b434.0	b429.0 b433.0	^b 429.0 ^c 433.0	b431.0 b434.0	b433.0 b438.0	b426.5 b429.0	b431.0 b434.0	b424.0 b427.0	b421.0 b424.0	b421.5 b425.0	b421.0 b421.5
441.	47.0 5 47.0 4 47.0 5	917	297.6 310.9 323.2		115 115 115	21/ 35 35/ 49 66/ 77	a410 0	DA 20 0	D417 0	b417.0	C414.0	D415-0	b416.5	D419.0	b413.0	D416-0	b413.0	D410.0	b415.0 b410.0 b414.0	D411.0

a_{Nonboiling}.

bSubcooled boiling.

Error

An error occurred while processing this page. See the system log for more details.

INSIDE DIAMETER BY 29.2-CENTIMETER-LONG TEST SECTIONS 230-50, 230-50B, 230-50C, AND 230-50E

 T_e = exit temperature, K; P_e = exit pressure, kN/m² abs; ΔP = pressure drop, kN/m².]

Run	i c	1 a	,, I T. 1	·			,	,			all temp	~	Тк				
<u>}</u> -		q efrom	T _i		P _e		اییا	L 25	11 47	. —				24 17	26.67	2: 94	28 ()
45		2292		360.9	١											1	
5E 6E	28.0	2723	316.5	360.9 360.9	689	a439.0	b453. b448.0 b453.0	b445.0	0446.0 0449.0	b448.0	5445.0 5450.0	6447.0 6450.0	b449.0 b453.0	b450.0 b450.0 b455.0 b458.0	b450.0 b453.0	b451.0 b453.0 b456.5 b456.5	b450.0 b453.0
/ F 8 E	1 28-0	3436 3782	304.3 298.7	360.9 360.9	687 687	a449.0 a454.0	b453.0 b456.0	b453.0 b455.0	b453.0 b455.0	0454.0 0458.0	0453.0 0456.0	6453.0 6455.0	b455.0 b459.0	b458.0 b460.0	b455.0 b459.0	b456.5 b462.0	b456.5 b460.0
1	l											2	2	ĺ	b	h	b
10£ 11£ 12F	28.1	1545 2320 2613	352.1 345.9	383.2 383.2 383.2	687	a426.5	a435.0	8447.0	a445.0	b450.0	0450.0	b451.0	b453.0	b452.0 b458.0 b456.5 b460.0 b461.5	t453.0	b453.0 b455.0 b458.0 b460.0	5451.0 5453.0
1 1 E	28.0	3152	341.5 332.1 325.9	383.2	687	a456.5	b458.0	6458.0 6460.0	b460.0	6460.0 6462.0	6459.0 6460.0	b458.0	5459.0 5460.0	b460.0	6457.0 6458.0	0460.0 0461.0	b457.0 b459.0
			32,00	303.12		1			i	l			!				
1 8F 1 9C	28.U 28.U	1582 2065 2355	369.3 360.9	394.3 394.3	688 688	a426.5 a439.0	a435.0	a437.0	a443.0 a454.0	a444.0 b455.0	a445.0	8446.0 0454.0	8448.0 6455.0	h450.0 b455.0 b458.0	6453.0	b451.5	0453.0
20E 21E 22F	28.0	12C36	355.9	394.3 394.8								10454.0	1 °455 - 0	U456.5	U455.0	0456.5	1 455.0I
23E	28.0 28.0 28.0	3152	350.4 343.7 339.8	394.3 394.3 394.3	687	a458.0	b458.0 b458.0 b460.0	b460.0	b461.0	6462.0	b460.0	0459.0 0458.0	b461.0 b460.0	0461.0 0462.0 0463.0 0450.0	0458.0 0460.0	1 450.0 1 452.0	0459.0 0461.0
25E	∠B. I	1532	370.4	394.3	688	a428.0	a434.0	a437.0	a443.0	a445.0	a445.0	a445.0	a448.0	b450.0	1449.0	451.0	t 450.0
2 he	28.U 28.0	1012	389.8	405.4	687	a424.0	a _{428.0}	a430.0	a434.0	a436.0	a435.0	a436.0	a440.0	a441.0	a441.0	a444.0	2441.D
29E 30F	28.0	1879	384.3 374.8	405.4	687 688	a433.0	a439.0	a441.5 b451.5	a446.0	1454.0	a448.0	8448.0 5452.0	8450.0 5454.0	a441.0 t 451.5 t 454.0	450.0	1454.0	1451.5 1454.0
31E 32F 33C	28.0 28.0		368.2	405.4	688	0454.0	0453.0	5454.0	0457.0	0455.0 0458.0	455.0	455.0	°455.0	1.458.0	457.0	456.5	459.0
7.30	78.0	3095	355.9	405.4	688	b454.0	b458.0	459.0	b461.0	1461.5	460.0	460.0	481.0	b463.0	459.0	452.0	~45U.U
48E 39E	47.0	2796 3593	333.7 324.8	360.9 360.7	687 687	8413.0 8429.0	a421.0	a424.0	a429.0 a447.0	3431.5 3448.0	1449.0	3433.0 8450.0	4438.0 8453.0	1441.0	4439.0 454.0	1444.0	8443.0
40F 41F	47.0	4224 4854	318.7	360.9 361.2	687 688	a445.0	a436.0 a454.0 a459.0	a456.5	a458.0	463.0	457.0	L 456.0	6460.0 6461.5	463.0 464.0 466.5	460.0	459.0 454.0 455.0	463.0
42E	47.0	5453	306.5	360.9			6463.0	463.0	°463.0	464.0	1461.5	460.0			,	470.0	455.0
44F	40.7	2449	359.3 354.8	383.2 383.2	683 686	a 420.0	a425.0 a435.0 a446.0 a456.5	a 428.0	a435.0	a436.0	1438.0	439.0	1443.0 8453.0	1447.3 1456.5	1446.0 455.0	1451.0 458.0	1 464 01
46t 476	46.7 47.0	3404	349.8	383.2	687	a 439.0	446.0 a456.5	a450.0	8456.0	458.0	456.5	456.5	459.0	462.0	458.0 461.0	452.0	458.0
48E	46.1	4602 5327	337.1	383.2	687	a 459.0	469.0	469.0	465.0	466.5	464.0	463.0	465.0	464.0 466.5 473.0	463.0	456.0	462.0
うい(.	47.0	2244	360.9	383.2	687	a 420.0	a425.0	a428.0	a431.0	a436.5	437.0	4437.0	3441+0	7443.€	1444.0	3448.0	4445.0
52c	47.0	2020	374.8	394.3	683	8424.0	a429.0	a428.0	a438.0	a440.0	A440.0	a440.0	A444.0	1447.0 1458.0 1461.5	2447.0	2449.0	1447.0
54E	47.0 47.0 47.0	3467	368.7 360.4 354.3	394.3 394.3 394.3	687	a446.0 a456.5	a455.0	a455.0	a458.0	459.0	457.0	456.5				1 451.5 1 454.0	458.0 461.0
26t	46.7	4350	350.9 347.1	394.3	687	a461.0	a464.0	465.0	467.0	1468.0	465.0	463-0	466.0	468.0	463.0	456.0	452.0
58E 59E	47.0	5012 5295	344.8 342.1	394.3	687	b465.0	0468.0	b469.0	1 469.0	1.469.0 469.0	467.0	465.0	1468.0 1467.0	469.0 470.0 470.0	465.0	459.0	465.0 465.0
buf	47.0	5548	339.6	394.3	687	465.0	466.0	467.0	L 466.5	t 468.0	466-0	1 464.0	1 466.5	470.0	465.0	470.0	466.5
62F	46.7	2355	382.6	405.4	689	a438.0	a444.0	² 448.0	a453.0	1454.0	454-0	453.0	t 458-0	1458.0	458.0	450.0	458.0
6 3F 6 4F	46.7 46.7 46.7	4C35	373.7 364.8	405.4 405.4 405.4	687	a450.0	460.0	461.0	463.0	463.0	461.5	460.0	461.5	464.0	461.0	450.0	458.0 458.0 461.5
666	47.0	5012	361.5 355.9 350.9	405.4	686 686	a 462.0	464.0	465.0	465.0	466.0	464.0	461.5	464.0	466.5	463.0	457.0	457.0 457.0 454.0 457.0
486	47.0	3051	375.9	405.7	687	A449.0	A458.0	a458.0	a460.0	461.5	460.0	458.0	1 460.0	461.0	458.0	450.0	457.0
716	47.3 47.0	1154	405.9	416.5	688	a431.5	a436.0	a437.0	a440.0	2441.5	2441.5	² 441.5	3443.0	1445.0	1445.0	1448.0	1446.0
77E	40.1	2389	398.2 393.7	416.5 416.5	687	a 450.0	8456.5	a458-0	7461.0	U461-0	459.0	1459.0	1 461.5	1 464.0	1 461.0	1 450.0	: 456.5
74F 75E	47.0	3972	387 • 1 377 • 6	416.5 416.8	687 687	h	0459.0 0463.0	D	here a	tiere o	1	1	heer c	1	11 / / 0 0	11/12	11110
76+ 77+ 78+	47.0 46.7 47.0	5012	370.9 366.5 363.7	416.5 416.5 416.5	687 687 687	0463.0 0466.0	0465.0 0468.0	5466.0 5467.0	1465.0 1466.0	5467.0	462.0	461.0	1 464.0	1 465.0 1 467.0 1 469.0	1 462.0	456.0	464.0
185	4 /	2292	303.1	415.5													
796 866	47.3	1655	417.6	427.6	687 687	a443.0 a450.0	a446.0 a454.0	a447.0 a454.0	a448.0	h449.0 b455.0	1448.0 0454.0	5448.0 5453.0	5449.0 5454.0	1 449.0 0454.0	1 448.0	1449.0 1453.0	b448.0 1451.5 b454.0 1456.0 b458.0 1459.0 b461.0 b464.0
81F 82E	47.1	2074 2443	407.9	427.6	687 687	a 454.0	a456.0	a 456.5 b 459.0	a458.0	0458.0 0461.0	1457.0	t 456.0 b 459.0	0456.5 0459.0	0456.0 6459.0	1 454.0 0 455.0	1 456.0 1 458.0	1 456.0
831	47.4 47.5 46.7	2516	399.8 392.6	427.6	688	0462.0 0461.5	0462.0 0463.0	0463.0 0464.0	6463.0	0464.0	460.0	£459.0	0460.0	0461.0 0461.5	1 459.0	0450.0 0451.5	1 459.0
85E 86E	47.0	4192 5012	386.5 378.2	427.6 427.6	688	5464.0 5467.0	5465.0 5467.0	5465.0 5468.0	b 466.0	5466.5	5464.0	0459.0 0461.0	5461.5 5464.0	6466.5	460.0	b454.0	0461.0
87E	2 a • r	816	404.0	416.2				a	a								
84F 89E	28.1	1636	390.9	416.5	688	a 448.0 a 458.0	a 453.0 b 458.0	a455.0 b458.0	b 455.0 b 459.0	0455.0 0459.0	b455.0 b458.0	b454.0 b457.0	0455.0 0457.0	0455.0 0458.0	6455.0	b455.0 b458.0	5453.0 5456.5 5451.0 5461.0
91+	28.1 28.1 28.1	2733 3688	372.6	416.5	688 688	0460.0 0461.5	0463.0 0464.0	0463.0 0464.0	0463.0 6464.0	D463.0	0462.0 0461.5	0461.0 1460.0	D462.0	0462.0 0464.0	0461.5 0461.0	D453.0	D451.0
476	k # * 1	3972	352 - 1	416.5	688	463.0	465.0	465.0	464.0	465.0	(463.0	461.0	°463.0	465.0	["461.0	0455.0	462.3

TABLE VI. - HEAT-TRANSFER AND PRESSURE DROP DATA FOR 0.584-CENTIMETER INSIDE DIAMETER BY 14.6-CENTIMETER-LONG TEST SECTION 230-25F $\begin{bmatrix} G = mass \ velocity, \ kg/(sec)(m^2); \ q = heat \ flux, \ kW/m^2; \ T_i = inlet \ temperature, \ K; \ T_e = exit \ temperature, \ K; \ P_e = exit \ pressure, \ kN/m^2 \ abs; \ \Delta P = pressure \ drop, \ kN/m^2. \end{bmatrix}$

Run	G	q	Ti	Т _е	Pe	ΔΡ							Axial o	listance	from sta	rt of hea	ting, z,	cm						
							0.635	1.27	1.68	2.54	3.17	3.81	4.44	5.71	6.98	8.25	9.52	10.79	12.06	12.70	13.33	13.97	14.10	14.29
														Inner-w	all temp	erature,	т _w , к							
5 6	3.54	501	351.2 342.1 296.5	368.2	114 114 114	<7 <7 ~7	D400.0	1397.0	0399.0	C395.0	D402.0	D402.0	D401-0	D402.0	b399.0	D398.0	b432.0	D402.0	b395.0	b396.5	D398.0	b395.0	b393.0 b400.0 b405.0	D400.0
10 12 13 14	7.05	224 372	293.2 350.9 351.5 347.3	314.3		< 7	a375.0	380.0 397.0	10397.0	a384.0	a385.0	a386.5	a388.0	a389.0 b402.0	a389.0	b390.0	b390.0 b401.5	b390.0 b401.5	b391.5	b390.0	D396.0	b390.0	b404.0 b392.0 b402.0 b403.0	b393.0
28	7.02	649	375.9	394.3	687	10	a433.0	a443.0	a445.0	b445.0	b445.0	1446.0	L446.0	447.0	b448.0	b448.0	b449.0	b449.0	b449.0	b448.0	b451.5	b446.5	b452.0	⁰ 451.5
169	23.1 28.0 24.0 28.1 28.0 28.1 23.1 23.1	3215 3215 3184 3184 3184 3184 3184 3184	293.2 298.2 301.5 303.4 307.1 310.9 313.2 315.1 318.7 320.9	324.3 327.6 329.8 333.4 337.1 339.3 340.9 344.3	112 112 112 112 112 112 112	13 12 12 13 14 13 14 14 14	£415.0 £414.0 £414.0 £413.0 £415.0 £414.0 £415.0	414.0 413.0 413.0 413.0 413.0 413.0 414.0	415.0 415.0 415.0 414.0 416.5 416.5 416.5	0416.5 416.0 416.0 416.5 416.5 416.5 417.0	b421.0 b420.0 b420.0 c419.0 b420.0 b421.5 b421.0	5420.0 5420.0 5419.0 5419.0 5419.0 5420.0 5420.0	2419.0 0420.0 1418.0 1418.0 1418.0 1418.0 1419.0 0420.0	b420.0 L419.0 b419.0 b419.0 c420.0 c420.0 c420.0 c419.0	0425.0 0421.0 0421.0 0419.0 0421.5 0421.5 0421.5	b419.0 b418.0 b418.0 b418.0 b418.0 b419.0 b419.0 b416.5	0418.0 0419.0 0419.0 0419.0 0420.0 0420.0	b420.0 b419.0 b419.0 b421.0 b420.0 b419.0 b419.0	b419.0 b419.0 b419.0 b419.0 b419.0 b419.0 b419.0	b419.0 b419.0 b418.0 b421.0 b418.0 b419.0 b418.0	b425.0 b425.0 b424.0 b425.0 b424.0 b422.0 b423.0 b421.5	b416.5 b416.5 b415.0 b416.5 b416.0 b415.0 b415.0	b424.0 b423.0 b423.0 b421.5 b422.0 b421.5 b420.0 b421.0 b421.0	b422.0 b422.0 b420.0 b420.0 b421.0 b419.0 b420.0
228 229 230 231 232 233 234 235 237 248 249 241 242 243 244 245 246 247 248	23.0 24.0 24.0 24.0 24.0 24.0 24.0 24.0 24	3184 3184 3215 3185 3215 3215 3215 3247 3247 3247 3247 3215 3215 3215 3215 3215 3215 3215	297.1 301.2 304.8 307.6 311.5 315.4 320.9 323.7 331.2 339.6 339.6 340.9 343.7 349.6 357.3 361.8 369.3 373.2	327.6 330.7 333.7 337.6 341.8 347.1 349.8 357.2 361.2 364.3 366.8 366.8 375.4 380.4 382.9 390.4 390.4	308 308 308 308 308 308 308 308 308 308	12 12 13 14 14 14 14 14 14 14 14 14/15 14/15 14/16 14/17 14/19 16/23 17/26 23/32 28/37 35/41	6436.0 6436.0 6436.0 6438.0 6439.0 6439.0 6449.0 6443.0	438.0 (438.0 (438.0 (438.0 (439.0 (443.0 (44	(438.0 (439.0 (440.0 (440.0 (440.0 (442.0 (442.0 (443.0 (444.0	L439.0 L439.0 L440.0 L440.0 L442.0 L443.0 L443.0 L445.0 L445.0 L445.0 L445.0 L445.0 D444.0 L443.0 D444.0 L443.0 L443.0	D440.0 L441.0 D442.0 D442.0 D443.0 D444.0 C444.0 C444.0 C444.0 D444.0 D444.0 D444.0 D444.0 D444.0	D440.0 D441.5 L441.5 L441.5 L443.0 D441.5 L443.0 D444.0 D446.0 D4	1441.0 1441.5 1442.0 1443.0 1443.0 1444.0 1445.0 1445.0 1445.0 1445.0 1445.0 1445.0 1445.0 1445.0 1445.0 1446.0	L441.5 P441.5 P441.5 P441.5 P441.5 P441.5 P441.0 P442.0 P445.0 P445.0 P445.0 P445.0 P445.0 P445.0 P445.0 P446.0 P466.0	D444.0 D444.0 D445.0 C445.0 C446.0 D446.0 D446.5 C446.5 D448.0 D448.0 C445.0 C445.0 C445.0 C445.0 C445.0 C445.0 C445.0	D444.0 D444.0 D444.0 D444.0 D445.0 D445.0 D445.0 C446.0 C446.0 C446.0 D447.0 D447.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	L444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D444.0 D445.0 D445.0 D445.0 D445.0 D446.0 D4	D445.0 D445.0 D445.0 D445.0 C445.0 C445.0 D446.0 C445.0 D446.0	0446.0 0446.5 0446.5 0446.5 0446.5 0446.5 0446.0 0447.0 0447.0 0448.0 0448.0 0448.0 0448.0 0448.0 0448.0 0448.0 0448.0 0448.0	0450.00 0450.00 0450.00 0450.00 0450.00 0450.00 0450.00 0450.00 0450.00 0451.00 05446.00	D443.0 D443.0 D443.0 D444.0 D443.0 D443.0 D441.5 D441.5 D441.5 D441.5 D441.5	5 b444.0 5 b445.0 6 b444.0 6 b442.0	0446.5 0447.0 0447.0 0447.0 0447.0 0447.0 0447.0 0447.0 0447.0 0447.0 0446.5

9.1	28.1 3184	304 8	330.9	488	12	8441.5 8450.0 3452.0 3453.0 3455.0 3455.0 3455.0 3455.0 3456.0 3457.0 3456.5 3456.0 3456.0 3456.0 3460.0 3456.0 3456.5
	23.1 3184				12	446.0 4452.0 456.0 456.0 455.0 455.0 457.0 457.0 458.0 459.0 5459.0 5459.0 5459.0 5458.0 5458.0 5462.0 5458
83	23.1 3184				12	8449.0 (453.0 0455.0 1455.0 1458.0 1458.0 1459.0 1459.0 0461.5 0460.0 0450.0 0457.0 0459.0 0460.0 0464.0 0459.0 0460.0 0461.0
84	28.1 3184				12	\$453.0 455.0 457.0 458.0 460.0 459.0 460.0 460.0 460.0 460.0 460.0 5463.0 \$463.0 \$463.0 \$465.0 \$460.0 \$462.0 \$466.0 \$466.0 \$465.
85	28.1 3184				12	L462.0 b462.0 b466.5 b461.0 b463.0 b465.0
86	23.1 3184		362.1		12	\$455.0 455.0 457.0 458.0 461.5 462.0 462.0 463.0 463.0 463.0 5463.0 5453.0 5460.0 5462.0 5461.0 5466.5 5461.5 5463.0 5466.0
87	28.1 3184		367.6		12	456.0 457.0 459.0 460.0 462.0 462.0 463.0 463.0 465.0 465.0 465.0 465.0 465.0 463.0 461.0 461.5 467.0 462.0 462.0 463.0 466.5
88	29.1 3184		372.1	688	12	458.0 458.0 460.0 461.5 463.0 462.0 463.0 464.0 465.0 5464.0 5464.0 5454.0 5461.5 5463.0 5462.0 5469.0 5463.0 5465.0 5468.0
AQ		352.3		688	12	458_0 459_0 467_0 467_0 467_0 464_0 464_0 464_0 464_0 464_0 464_0 5466_0 5466_0 5464_0 5464_0 5468_0
90			383.2		12	460.0 461.0 463.0 463.0 465.0 465.0 465.0 465.0 465.0 465.0 466.5 465.0 465.0 465.0 4640.0 464.0 464.0 464.0 463.0 466.0 463.0 466.5
91	25.3 3184			688	12	460.0 460.0 463.0 463.0 465.0 465.0 465.0 465.0 465.0 465.0 465.0 465.0 465.0 463.0 463.0 463.0 469.0 469.0 463.0 468.0
92	28.3 3184			689	12	461-5 462-0 464-0 464-0 465-0 465-0 465-0 466-0 465-0 5466-5 5465-0 5455-0 5453-0 5464-0 5463-0 5470-0 5463-0 5466-0 5469-0
93	23.0 3184	367.6	393.2		12/13	467.0 462.0 464.0 464.0 465.0 465.0 465.0 465.0 465.0 5465.0 5465.0 5465.0 5463.0 5463.0 5463.0 5469.0 5463.0 5465.0 5463.0 5465
94	23.0 3184				12	\$463.0 \$464.0 \$465.0 \$465.0 \$467.0 \$466.0 \$467.0 \$466.5 \$469.0 \$466.0 \$466.0 \$464.0 \$4
95			402.6		12/14	3465.0 3465.0 467.0 467.0 468.0 467.0 467.0 467.0 467.0 468.0 468.0 466.5 5456.0 5468.
96	23.3 3184				12/15	5465-0 465-0 467-0 467-0 466-0 467-0 467-0 467-0 467-0 468-0 467-0 5456-5 5463-0 5464-0 5463-0 5467-0 5465-0 5466-5 5470-0
97	23.0 3184				12/16	465.0 465.0 466.5 466.0 467.0 466.0 466.0 4666.5 467.0 466.5 467.0 466.5 5456.0 5463.0 5463.0 5463.0 5469.0 5465.0 5467.0 5469.0
98	23.3 3184		412.1		13/17	\$465.0 \$465.0 \$466.0 \$466.5 \$467.0 \$467.0 \$467.0 \$466.5 \$467.0 \$465.0 \$455.0 \$463.0 \$464.0 \$463.0 \$469.0 \$464.0 \$464.0 \$466.5 \$467.0 \$466.5 \$467.0 \$466.5 \$467.0 \$466.5 \$467.0 \$466.5 \$467.0 \$4
99	23.0 3184		413.7		14/17	\$465.0 \$465.0 \$466.0 \$466.0 \$466.5 \$466.5 \$466.0 \$466.0 \$466.5 \$467.0 \$465.0 \$454.0 \$462.0 \$463.0 \$462.0 \$469.0 \$464.0 \$464.0 \$466.0 \$469.0
100	23.0 3184		417.6		16/19	\$465.0 (465.0 (466.0 (4
101	23.1 3184		420.1		17/22	\$465.0 \$465.0 \$466.0 \$466.0 \$466.0 \$465.0 \$465.0 \$464.0 \$466.0 \$465.0 \$454.0 \$461.5 \$5463.0 \$463.0 \$468.0 \$5463.0 \$5465.0 \$546
102			422.6		18/25	\$466-0 \ 466-0 \ 466-0 \ 466-0 \ 466-0 \ 465-0 \ 465-0 \ 465-0 \ 464-0
103	24.0 3184		430.4		27/30	3465.0 466.0 466.0 466.0 466.0 465.0 465.0 465.0 466.0 465.0 466.0 465.0 466.0 5463.0 5461.0 5461.0 5461.0 5467.0 5467.0 5467.0 5465.0 5465.0 5465.0
104	24.0 3184		427.1		24/28	5466-0-5466-0-5466-0-5466-0-5466-0-5465-0-5465-0-5464-0-5465-0-5464-0-5463-0-5461-0-5462-0-5467-0-5463-0-5463-0-5467-0-5463-0-5460-0-5465-0-5460-0-5465-0-5460-0-54
105	28.1 3215		403.2		12/14	^{b471.5} ^{b465.0} ^{b471.0}
106	28.1 3184		401.5		12/14	463.0 464.0 466.0 466.0 466.0 466.0 466.0 466.0 466.0 466.0 466.0 465.0 465.0 465.0 465.0 467.0 467.0 467.0
107	23.1 3184		399.8		12/14	\$463.0 463.0 465.0 465.0 467.0 466.0 466.0 466.0 466.0 466.0 466.0 5466.
108	24.0 3184	355 - 1			12	\$460.0 \$460.0 \$463.0 \$463.0 \$464.0 \$465.0 \$465.0 \$465.0 \$466.0 \$465.0 \$465.0 \$464.0 \$464.0 \$464.0 \$464.0 \$4670.0 \$4670.0
109	28.4 3215				12	\$\\\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \
110			337.1		12	5 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
111			343.7		12	\$\\\\ 0\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\
112	29.0 4823				12	5466.5 467.0 468.0 468.0 470.0 469.0 470.0 470.0 470.0 474.0 474.0 5474.5 5469.0 5471.0 5471.0 5473.0 5478.0 473.0 5478.0 5478.0
113	23.0 4854				12	\$466.5\$468.0\$469.0\$469.0\$470.0\$470.0\$470.0\$470.0\$470.0\$474.0\$474.0\$470.0\$472.0\$459.0\$471.5\$\$473.0\$473.0\$478.0\$\$472.0\$472.0\$475.0\$478.0\$
114	28.3 4791				12	5448.0 5468.0 5469.0 5469.0 5470.0 5470.0 5471.0 5471.0 5473.0 5473.0 5471.0 5459.0 5470.0 5471.5 5478.0 5471.0 5474.0 5478.0
115	28.2 4791		357.6	688	12	\$468.0 \$469.0 \$469.0 \$471.0 \$471.0 \$470.0 \$471.0 \$470.0 \$473.0 \$471.0 \$472.0 \$472.0 \$459.0 \$471.0 \$471.5 \$478.0 \$471.0 \$471.0 \$475.0 \$4779.0
116	28.1 4791		362.9	688	12/13	~ 440 0 440 0 470 0 440 0 471 0 471 0 471 0 471 5 471 0 474 0 474 0 471 5 471 5 471 5 471 0 471
117	28.0 4854		365.1	688	12/13	1/469 0 5469 0 471 0 5471 0 5471 0 5471 5 5471 5 5471 5 5474 0 5473 0 5472 0 5470 0 5471 0 5474 0 5479 0 5471 0 5474 0 5471 0 5474 0 54
118	28.0 4823		367.6		13	469.0 469.0 471.0 470.0 471.5 471.5 471.5 471.5 472.0 474.0 472.0 471.5 470.0 471.0 472.0 472.0 472.0 479.0 479.0
119	28.0 4823		371.8		13/14	5449 0 5449 0 5470 0 5470 0 5471 5 5471 0 5471 5 5471 5 5471 5 5474 0 5471 5 5471 0 5469 0 5471 0 5471 0 5471 5 5471 0 54
120	23.0 4791		374.3		13/14	5469.0 470.0 470.0 469.0 471.0 471.0 471.0 471.5 471.0 474.0 474.0 471.5 471.0 469.0 470.0 471.5 471.5 471.0 474.0 474.0 474.0
121	28.0 4854		378.2		12/14	- Unio o Caro o Caro o Caro o Caro s Caro s Caro s Caro s Caro s Caro o Caro o Caro o Caro o Caro o Caro o Caro
122	28.3 4823		379.8	687	13/14	- 5440 0 5440 0 5471 5 5470 0 5473 0 5473 5 5471 5 5471 5 5474 0 5472 0 5471 5 5459 0 6470 0 6471 5 5476 5 6476 5 6470 0 6474 0 6474 0 6477 0
123	24.0 4791		382.6		13/14	\$470.0 \ 470.0 \ 471.5 \ 471.5 \ 472.0 \ 471.5 \ 472.0 \ 473.0 \ 473.0 \ 475.0 \ 475.0 \ 473.0 \ 473.0 \ 470.0 \ 471.0 \ 473.0 \ 471.0
124	28.0 4886	348.7	387.6	687	13/15	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
125	28.0 4823		392.9		14/16	473.0 473.0 474.0 473.0 475.0 475.0 475.0 475.0 475.0 476.5 0476.5 0476.0 0470.0 0470.0 0471.5 0474.0 0471.0 0470.0 0476.0 0476.0
126	23.1 4823	356.2	394.8	687	12/18	471.5 471.5 474.0 474.0 475.0 475.0 475.0 475.0 476.5 474.0 0476.5 474.0 0470.0 0470.0 0471.5 0474.0 0471.5 0476.0 0470.0
127	28.1 4854	359.6	398.2	687	11/21	\$474.0 \$474.0 \$475.0 \$475.0 \$474.0 \$475.0 \$476.0 \$475.0 \$476.5 \$476.0 \$474.0 \$470.0 \$471.5 \$474.0 \$476.5 \$476.5 \$476.0 \$476.5 \$4
128	28.1 4823	364.3	402.9	689	13/20	473.0 6473.0 6473.0 6473.0 6475.0 6473.0 6475.0 6475.0 6475.0 6475.0 6476.0 6473.0 6470.0 6470.0 6473.0 6473.0 6473.0 6470.0 647
129	28.3 4791	367.3	405.4	688	10/21	474.0 474.0 475.0 474.0 475.0 475.0 475.0 475.0 475.0 475.0 475.0 471.0 0469.0 0470.0 0473.0 0473.0 0471.5 6474.0 0479.0
130	28.3 4791	370.9	408.7	687	12/24	6474.0 6474.0 6475.0 6474.0 6475.0 6474.0 6475.0 6475.0 6475.0 6473.0 6469.0 6471.0 6473.0 6473.0 6473.0 6479.0
131	28.3 4791	373.2	410.9	687	12/26	674.0 474.0 475.0 475.0 5474.0 5475.0 5474.0 5475.0 5474.0 5475.0 5476.0 5476.0 5470.0 5476.0
132	28.1 4854	369.8	408.2	687	10/26	475.0 475.0 477.0 477.0 477.0 477.0 477.0 475.0 475.0 9476.5 475.0 0474.0 0471.5 0473.0 0473.0 0473.0 0473.0 0473.0
133	2d.1 4791	364.8			12/22	475.0 475.0 477.0 475.0 5478.0 5475.0 5476.5 5475.0 5476.0
134	28.1 4823	353.2	391.5	687	13/17	1474.0 6475.0 6476.5 6476.5 6478.0 6476.5 6478.0 6476.5 6477.0 6476.5 6475.0 6475.0 6476.5 6476.5 6478.0 6477.0 6476.6 6477.0 64
135	28.3 4791				14/15	1475.0 1475.0 1477.0 1478.5 1478.0 0478.0 1478.0 1478.0 0480.0 0478.0 0475.0 0476.0 0478.0 0480.0 0475.0 0480.0
136	28.0 4823	332.1	370.9	688	12/13	£474.0 £474.0 £476.5 £475.0 £477.0 £476.5 £478.0 £477.0 £478.0 £478.0 £478.0 £476.5 £476.5 £476.5 £476.0 £476.0 £476.5
						by the second of
303	28.3 4791				12/21	\$\\\ \frac{6472.0}{6472.0} \\ \frac{6473.0}{6474.0} \\ \frac{6473.0}{6475.0} \\ \frac{6475.0}{6475.0} \\ \frac{6475.0}{6475.0} \\ \frac{6475.0}{6475.0} \\ \frac{6475.0}{6476.0} \\ \frac{6475.0}{6476.0} \\ \frac{6476.0}{6476.0} \\ \frac{6476.0}{64
304	28.1 4791	369.8	407.6		12/24	6473.0 6473.0 6473.0 6473.0 6474.0 6474.0 6474.0 6475.0 6475.0 6476.0 64
305	28.3 4791	374.3			12/28	6472.0 6474.0 6474.0 6475.0 6475.0 6473.0 6473.0 6473.0 6473.0 6474.0 6472.0 6474.0 6471.0 6472.0 6475.0 64
306	28.1 4791	376.5	414.3		12/29	~472.0 ~472.0 ~472.0 ~474.0 ~473.0 ~473.0 ~473.0 ~472.0 ~414.0 ~472.0 ~474.0 ~472.0 ~4
307	28.3 4760	379.3	417.1		14/31	~413.01 ~413.07 ~414.01 ~473.01 ~413.01 ~413.01 ~410.01 ~412.0
308	28.3 4791	381.5	419.0		14/34	1 4/1.5 4/1.5 4/2.0 4/1.5 4/2.0 4/1.5 4/2.0 4/0.0 4/0.0 4/0.0 4/1.5 4/1.
309	28.0 4823	387.6	424.8		21/48	472.0 471.5 474.0 471.5 473.0 471.5 473.0 471.5 473.0 473.0 473.5 473.0 473.5 473.0 473.5 473.6 473.5 473.6
310	28.3 4791	392.1	429.3		31/45	\$\begin{array}{cccccccccccccccccccccccccccccccccccc
311	28.1 4823	394.0			34/48	74/5.0] - 4/1.5] - 4/
312	28.3 4823	397.1	433.7	688	43/50	-415-0 -417-5 412-0 417-0 417-5 -417-5 -417-5 -417-5 417-5 -417-5 -417-5 -417-5 417-5 417-5 417-5 417-5 417-5

aNonboiling.

b_{Subcooled boiling.}

TABLE VII. - HEAT-TRANSFER AND PRESSURE DROP DATA FOR 0.584-CENTIMETER INSIDE DIAMETER BY 14.6-CENTIMETER-LONG TEST SECTIONS 230-25, 230-25D, AND 230-25D

[G = mass velocity, kg/(sec)(m²); q = heat flux, kW/m²; T_i = inlet temperature, K; T_e = exit temperature, K; P_e = exit pressure, kN/m² abs; ΔP = pressure drop, kN/m².]

Run	G	q	Тi	Т _е	Pe	ΔΡ						Inner-w	all temp	erature,	T _w , K					
Ax	cial dist	ance fr	om start	of heati	ng, z,	cm	0.635	1.94	3.17	4.44	5.71	6.98	8.25	9.52	10.79	12.06	12.70	13.30	14.04	13.97
3 4 5 6 7 8 9	28.1 28.4 28.1 28.7 28.1 28.1 28.1	2783 4129 4570 4823 4949 5043	293.2 293.7	316.5 327.6 331.5 333.7	115 115	<7 <7 <7 <7 <7 <7 <7 <8 <8 <8 <8 <8 <8 <8 <8 <8 <8 <8 <8 <8	a416.5 b419.0 b420.0 b422.0 b423.0 b424.0	0435.0 0419.0 0420.0 0421.0	D431-5		b403.0 b414.0 b416.0 b418.0 b419.0 b420.0 b420.0	b408.0 b416.5 b415.0 b419.0 b420.0 b421.0 b421.0	b409.0 b418.0 b419.0 b419.0 b421.5 b423.0 b423.0 b423.0	b409.0 b416.5 b414.0 b415.0 b416.0 b474.0 b419.0 b420.0	b410.0 b418.0 b419.0 b419.0 b420.0 b420.0 b421.0 b421.5	b411.0 b418.0 b416.5 b418.0 b419.0 b419.0 b420.0 b420.0	b409.0 b416.5 b414.0 b413.0 b415.0 b417.0 b418.0 b418.0	b409.0 b416.0 b412.0 b413.0 b414.0 b416.0 b417.0	b408.0 b414.0 b413.0 b415.0 b416.0 b418.0 b418.0 b419.0	b408.0 b415.0 b415.0 b416.0 b418.0 b419.0 b421.0
11 12 13 14 15 c16	28.4 28.1 28.1	4066 4224 4413 4602	312.6 313.2 313.2 313.2	344.3 345.9 347.6 348.7 350.9 352.6	115 115 115 115	6 8 11 13 15	0427.0 0425.0 0426.0 0426.0 0425.0 0426.0	b424.0 b422.0 c424.0 b424.0 c425.0 c426.0	b421.0 b420.0 b423.0 b423.0 b424.0 b424.0		b419.0 b419.0 b420.0 b421.0 b421.0 b420.0	b417.0 b419.0 b420.0 b419.0 b420.0 b420.0 b420.0	b420.0 b420.0 b422.0 b422.0 b422.0 b423.0	b416.5 b417.0 b417.0 b418.0 b419.0 b421.0	b418.0 b419.0 b419.0 b420.0 b420.0 b421.5 b423.0	b416.0 b417.0 b420.0 b420.0 b423.0 b423.0	b414.0 b416.0 b419.0 b419.0 b420.0 b420.0	b415.0 b417.0 b419.0 b419.0 b420.0 b420.0	b415.0 b416.5 b418.0 b420.0 b420.0 b420.0	b416.0 b418.0 b419.0 b420.0 b422.0 b421.1
18 19 20 21 22 23	27.8 28.4 27.6 28.1 28.1	3184 3530 3940 4224 4602	331.5 330.9 331.5 330.9 330.9	364.8 368.2	115 115 115 115 115	12 16 20 26 30 36	t429.0 t428.0 t429.0 t429.0 t429.0	b425.0 b426.5 b428.0 b428.0 b428.0	b413.0 b419.0 b420.0 b423.0 b422.0 b426.5		b420.0 b419.0 b420.0 b420.0 b425.0	b421.0 b419.0 b422.0 b423.0 b427.0	b423.0 b423.0 b424.0 c425.0 b428.0	b419.0 b418.0 b421.0 b420.0 b424.0	b421.0 b421.5 b421.0 b421.5 b424.0	b420.0 b420.0 b421.0 b422.0 b425.5	b416.5 b415.0 b421.5 b418.0 b421.5	b419.0 b417.0 b417.0 b418.0 b421.5	b416.0 b415.0 b416.0 b416.5 b419.0	b419. b417. b418. b419. b421.
	_			t of heat	1	1	0.61	1.90	3.17 b412.0	4.44	b. 15	6.98	8.25	9.52					13.39	
28 48 58 68	28.7 45.9 45.9 45.9		295.4	322.1 315.4 328.2 329.8	115 115	10 12 16 17	6404.0 6425.0 6425.0	5400.0 5424.0 5425.0	b402.0 b425.0 b426.0	b424.0 b425.0	6401.5 6425.0 6426.0	t402.0 t425.0 t426.5	b400.0 b420.0 b423.0	6401.0 6423.0 6424.0	0401.0 0420.0 0421.0	b399.0 b416.5 b418.0	6430.0 6419.0 6421.0	b401.5 b421.5 b424.0	b412.0 b401.5 b420.0 b422.0	b403. b421. b423.
88 108 118 128 148	45.6 45.9 45.9	6146 3814 4570 4066 4507	304.3 323.7 323.7 330.4 324.3	343.2 346.5 350.9	115 115 115	17 14 19 23 19	6425.0 6422.0 6426.5	b422.0	b426.0 b421.5 b421.0 b426.0 c422.0	b418.0 b424.0	5416.5 5416.0	D414.0 D416.5 D421.5	0414.0 0415.0 0421.0	b414.0 b416.0 b420.0	0412.0 0414.0 0418.0	b410.0 b411.0 b415.0	b412.0 b413.0	0414.0 0415.0 0419.0	b414.0 b416.0	b415. b414.
168 178 188 198 208 218 228	28.4 28.2 28.2 28.2 28.2 28.2	4696 5138 5737 5989 6462 6903 6934	294.8 294.8 294.8 294.8 294.8	341.5	308 306 308 308 308	10 10 11 12 10/14 10/16 10/18	5444.0 5446.5 5448.0	5445.0 5446.0 5448.0	b448.0 b446.0 b448.0 b448.0 b448.0 b447.0 b448.0	b446.0 b446.0 b448.0	b446.0 b446.5 b448.0	b446.5 b446.5 b448.0	b444.0 b444.0 b446.5	0446.0 0445.0 0448.0	b445.0 b444.0 b446.5	5441.5 5443.0 5445.0	b444.0 b445.0 b445.0	5445.0 5446.0 5447.0	b448.0 b449.0 b449.0	b447. b449. b449.
248 258 268	28.4	6430	310.9	344.3 362.6 364.8	310	10 10/17 0/21	0441.5 0448.0 0446.0	1441.0 15446.5 15446.0	b439.0 b448.0 b447.0	6438.0 6445.0 6445.0	6439.0 15448.0 15446.0	b439.0 b448.0 b444.0	0436.5 0444.0 0445.0	. b438.0 - 0446.0 - 0444.0	b436.0 b444.0 b442.0	b435.0 b443.0 b443.0	0435.0 0444.0 0443.0	b436.5 b445.0 b443.0	b437.0 b446.5 b445.0	b438.
28B 2 9B				357.6 370.4		10 11/15	5430.0 5445.0	⁰ 428.0	b428.0 b444.0	b428.0 b443.0	6429.0 6444.0	D428.0	b427.0 b442.0	6428.0 6443.0	b427.0 b441.0	b425.0 b440.0	b428.0 b441.0	b429.0 b441.5	b431.0 b443.0	⁰ 430 ⁰ 442

31B 32B			353.7 352.6	385.4 387.1	310 310	6/22 3/31	b436.5 b439.0	^b 436.5 ^b 439.0	b436.5 b439.0	^b 436.0 ^b 438.0	b437.0 b439.0	^b 436.5 ^c 439.0	b435.0 b436.5	b435.0 b437.0	b434.0 b435.0	b433.0 b435.0	b434.0 b436.0	b435.0 b437.0	b436.5 b437.0	b436.0 b439.0
338 348 358	46.0 45.9 45.9	6745		345.4 353.2 352.6	115 115 115	22 41/55 41/55	b426.5	^b 436.0 ^b 424.0 ^b 424.0	b425.0	b423-0	b425.0	b425.0	b423.0	b424.0	b423.0	b422.0	b422.0	b421.0	b422.0	b421-0
378 388	14.3		294.8 294.3	322.6 347.1	114 114	8 10	b416.0 b404.0	b404.0 b406.0	b404.0 b408.0	b406.5 b409.0	b406.5 b409.0	b406.5 b410.0	b406.5 b408.0	6407.0 6408.0	b407.0	b406.5	b ₄ 38.0	b _{408.0}	b404.0 b410.0	b409.0
39B	45.9	5453	312.1	338.7	115	17	b423.0	b416.0	b417.0	b415.0	b415.0	b414.0	b414.0	b415.0	b413.0	b414.0	b414.0	b414.0	b416.0	b415.0
	xial dist	ance fr	om start	t of heat	ing, z,	cm		1.90	3.14	4.44	5.71	6.98	8.25	9.52				13.28		
3C 4C 5C 6C 7C 8C	28.7 28.7 28.1 28.4	4917 4570 3940 3688 2957	294.3 305.4 314.3 323.2 332.1	334.8 341.5 345.9 350.9 355.4 373.7	114 115 115 115 115	6 8 8 8 8 37/41	5415.0 5419.0 5420.0 5425.0	b408.0 b411.0 b412.0 b410.0 b411.0 b411.0	b411.0 b411.5 b411.0 b408.0	b414.0 b412.0 b410.0 b410.0	b408.0 b407.0 b406.5 b406.0	b411.0 b409.0 b406.5 b408.0		0411.0 0410.0 0408.0 0410.0	b406.5 b406.0 b405.0 b407.0	0409.0 0409.0 0405.5 0409.0	b410.0 b429.0 b429.0 b429.0	b412.0 b411.5 b409.0 b413.0	b419.0 b417.0 b415.0 b415.0	b412.0 b411.5 b409.0 b409.0
25C 26C	105.0		302.6 313.2	324.8 332.6		28/34 28/36	^b 413.0 ^b 416.0	b410.0 b413.0	b408.0 b412.0	b412.0 b416.0	b409.0 b413.0	b414.0 b417.0		b409.0 b413.0	^b 407.0 ^b 415.0	b409.0 b413.0	^b 438.0 ^b 413.0	b410.0 b414.0	^b 408.0 ^b 412.0	b407.0 b411.5
9C 14C 16C 18C 21C	70.3 70.3 70.6 70.9 70.5	5579 5674 4854		327.6 343.2 350.9 355.4 368.7		23 19 53 56 77	0415.0 0418.0 0409.0	b423.0 b408.0 b409.0 b408.0 b418.0	0407.0 0409.0 0406.5	0410.0 0409.0 0408.0	6406.0 6405.0 6404.0	0408.0 0408.0		b409.0 b409.0 b409.0	b406.5 b406.5	b408.0 b408.0	b406.0 b406.0	D421.5 D410.0 D408.0 D407.0 D414.0	b413.0 b413.0 b409.0	b409.0 b405.0 b405.0
A	xial dist	tance fr	om star	t of heat	ing, z,	cm	0.61	1.90	3.12	4.48	5.71	6.96	8.25	9.52	10.79					13.97
	1.74.3 1.73.6 103.6	4917	338.2	335.9 349.3 359.8	119	24/28 21/22 47/49	^a 412.0 ^a 393.0 ^a 394.0	^a 416.0 ^a 401.5 ^a 402.0	b416.5 b403.0 a404.0	^b 416.5 ^b 399.0 ^b 408.0	b418.0 b409.0 b409.0	5415.0 5409.0 5409.0	b415.0 b409.0 b409.0	b413.0 b409.0 b408.0	b413.0 b410.0 b408.0	b413.0 b411.0 b409.0	b414.0 b413.0 b438.0	b414.0 b413.0 b408.0	b 412.0 b 410.0 b 407.0	b406.5 b405.0 b403.0
120 160 220	27.4 28.2 28.3	6777 5327 3908	291.5 333.2 385.9	347.1 375.4 415.9		<7 0/10 10/34	b447.0 b445.0 b439.0	b446.5 b444.0 b438.0	b4446.0 b4444.0 b436.5	b448.0 b445.0 b437.0	b449.0 b445.0 b436.5	b447.0 b444.0 b436.0	b448.0 b444.0 b435.0	b446.5 b443.0 b434.0	b446.5 b442.0 b434.0	b447.0 b443.0 b434.0	b450.0 b444.0 b435.0	b451.0 b444.0 b436.5	b 455.0 b 444.0 b 436.5	b445.0 b440.0 b435.0
24D 30D	14.7 14.7			344.3 402.1		<7 0/12	^E 437.0 ^b 439.0	^b 437.0 ^b 439.0	^b 436.0 ^b 438.0	b436.5 b439.0	^b 437.0 ^b 438.0	^b 435.0 ^b 438.0	b436.5 b438.0	b436.0 b436.0	b435.0 b436.0	^b 437.0 ^b 436.5	b438.0 b437.0	^b 439.0 ^b 438.0	b 439.0 b 436.5	b 438.0 b 436.0
310 32D	14.4	895 3184	295.4 295.9	309.3 345.9		<7 <7	^a 356.5 ^a 398.0	² 374.0 ² 399.0	^a 383.0 ^a 401.5	^a 387.0 ^a 403.0	⁸ 391.5 ⁶ 403.0	²³ 92.0 ℃02.0	² 394.0 ² 402.0	² 395.0 ² 403.0	²³ 96.5 ² 402.0	² 397.0 ² 403.0	^b 398.0 b401.5	b398.0 b404.0	b 398.0 b 405.0	b395.0 b401.0
37D 38D 47D	14.4 14.7 14.4	3133 4823 3111	295.9 296.5 389.3	345.9 372.1 434.3	689 689 687	6 8/10 12/17	^b 450.0 ^b 456.0 ^b 453.0	⁶ 450.0 ⁶ 456.5 ⁶ 451.5	b450.0 b457.0 b451.5	^b 450.0 ^b 458.0 ^b 451.5	5450.0 5458.0 5451.5	5449.0 5457.0 5450.0	^b 451.0 ^b 457.0 ^b 450.0	6451.5 6456.5 6450.0	b451.5 b456.5 b450.0	b453.0 b458.0 b450.0	b454.0 b459.0 b451.0	b455.0 b459.0 b451.0	b 454.0 b 456.5 b 448.0	b452.0 b455.0 b449.0
48D 56D	28•2 28•2	8353 4381	297.1 399.8	364.3 433.2	687 687	9/13 31/38	² 461.5 ⁰ 454.0	¹ 465.0 ¹ 454.0	^b 464.0 ^b 453.0	⁰ 467.0 ⁰ 454.0	⁰ 467.0 ⁰ 454.0	0465.0 0453.0	b465.0 b453.0	0465.0 0453.0	b465.0 b453.0	b465.0 b454.0	1456.5 1455.0	b466.0 b455.0	D464.0 D452.0	b460.0 b451.0
57D	141.2	11410	306.5	325.9	123	38/55	² 419.0	⁶ 421.5	b421.5	b425.0	b426.5	b423.0	b424.0	b422.0	b422.0	b422.0	D424.0	b423.0	b 421.5	b412.0

^aNonboiling.
^bSubcooled boiling.
^cAutomatic shutdown after data taken.

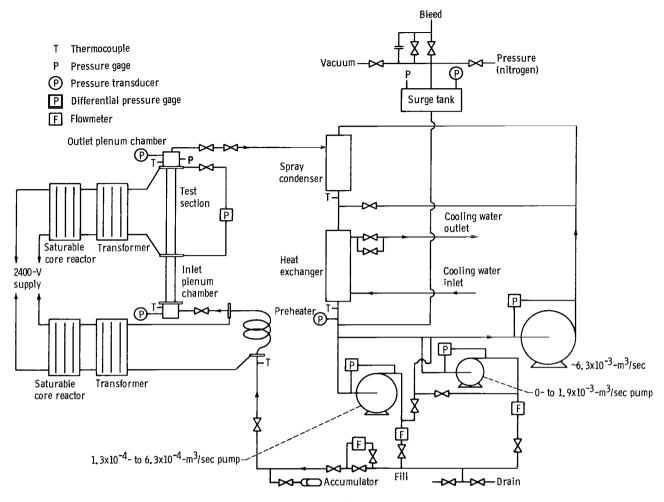


Figure 1. - System flow diagram.

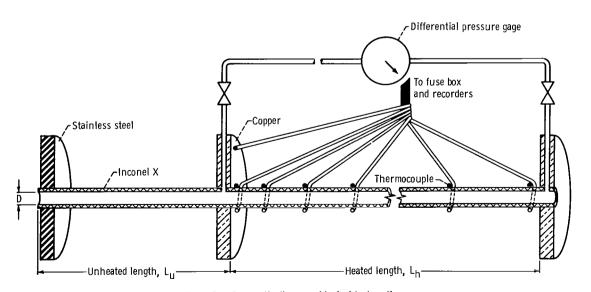


Figure 2. - Schematic diagram of typical test section.

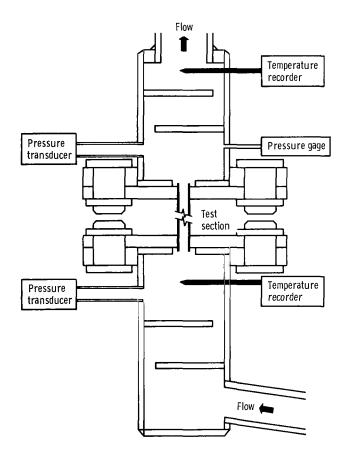


Figure 3. - Plenum chambers.

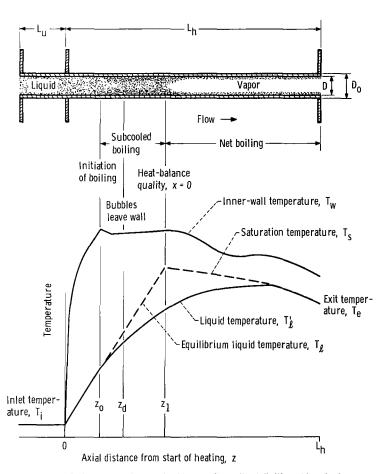


Figure 4. - Typical temperature and voidage profiles with definition of terminology.

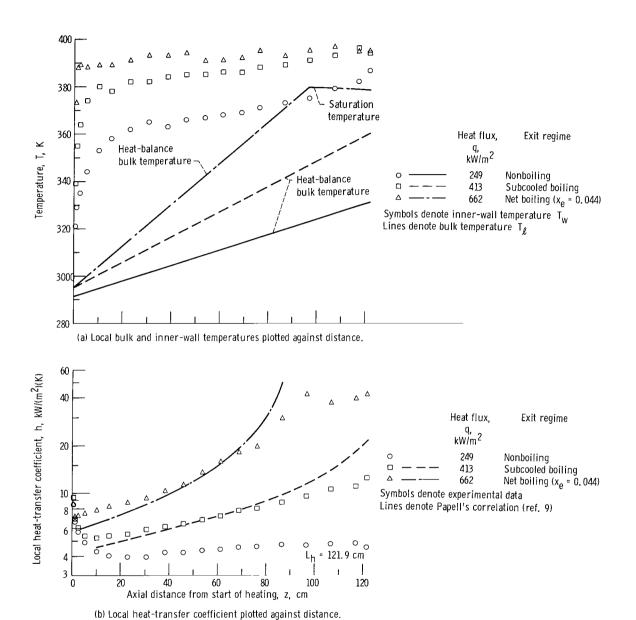


Figure 5. – Typical heat-transfer data for large subcooling at inlet (~83 K). Tube inside diameter D = 1.219 centimeters; mass velocity $G \approx 5.9 \text{ kg/(sec)(m}^2)$; exit pressure $P_e \approx 117 \text{ kN/m}^2$ abs.

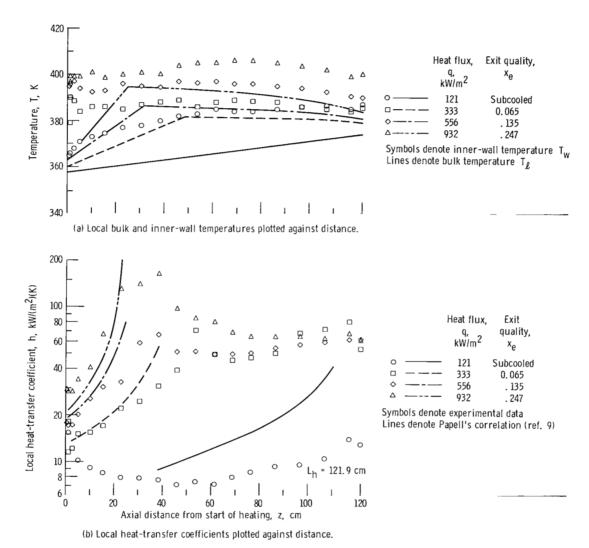


Figure 6. – Typical heat-transfer data for small subcooling at inlet (~22 K). Tube inside diameter D = 1.219 centimeters; mass velocity $G \approx 5.9 \text{ kg/(sec)(m}^2)$; exit pressure $P_e = 119 \text{ to } 146 \text{ kN/m}^2 \text{ abs}$.

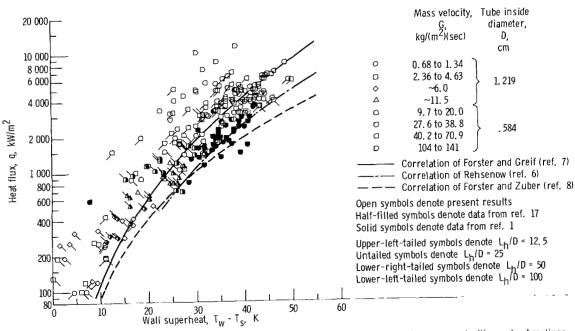


Figure 7. - Heat flux plotted against wall superheat during subcooled boiling for various mass velocities and subcoolings. Exit pressure $P_e \approx 114 \text{ kN/m}^2$ abs.

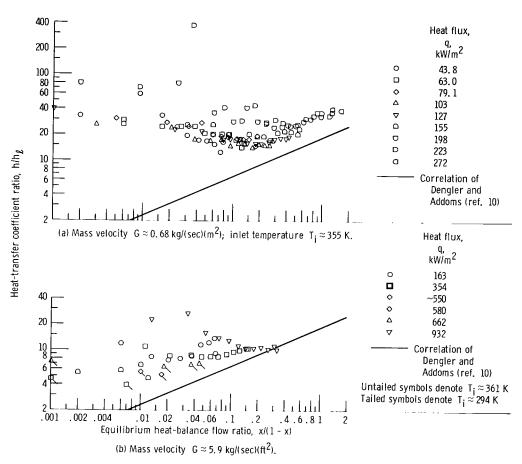


Figure 8. - Ratio of boiling to liquid heat-transfer coefficients plotted against equilibrium heat-balance flow ratio x/(1 - x). Test section 480-100M (table I); exit pressure $P_e \approx 117 \text{ kN/m}^2$ abs.

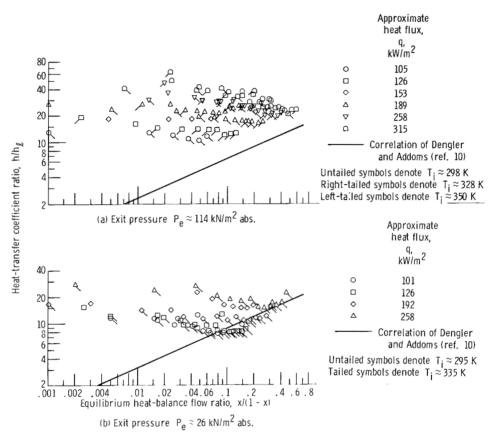


Figure 9. - Ratio of boiling to liquid heat-transfer coefficients plotted against equilibrium heat-balance flow ratio x/(1-x). Test section 480-50M (table II); mass velocity $G\approx 0.68$ kg/(sec)(m²).

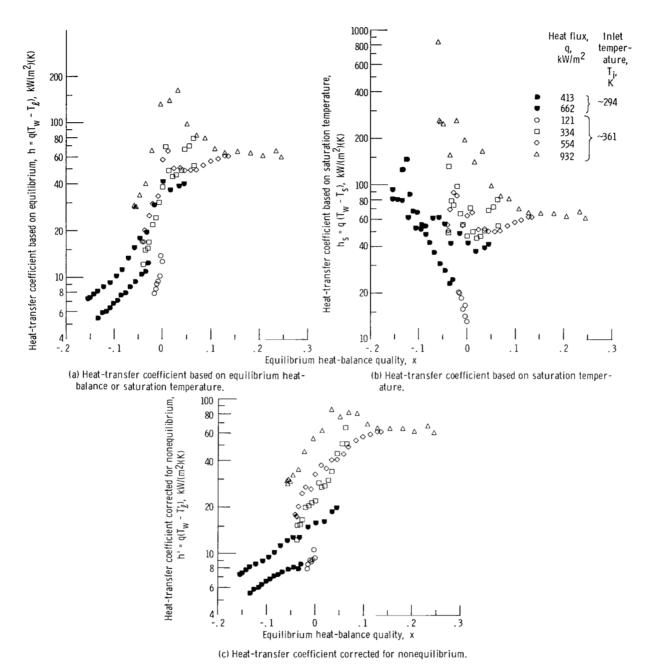


Figure 10. - Heat-transfer coefficient plotted against equilibrium heat-balance quality for various methods of calculating heat-transfer coefficient. Test section 480-100M (table I); mass velocity $G \approx 5.9 \text{ kg/(sec)(m}^2)$; exit pressure $P_e \approx 119 \text{ to } 146 \text{ kN/m}^2 \text{ abs.}$

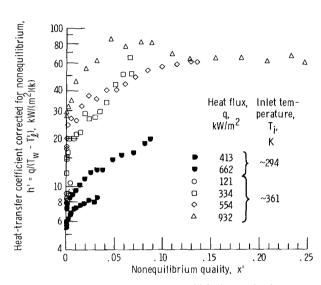


Figure 11. – Heat-transfer coefficient corrected for non-equilibrium plotted against nonequilibrium quality. Test section 480–100M (table I); mass velocity G ≈ 5.94 kg/(sec)(m²); exit pressure $P_e\approx 119$ to 146 kN/m² abs.

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